

Performance of a Porous Microchannel Heat Sink

This model studies the performance of a microchannel heat sink (MCHS) with a porous block structure and compares its performance with that of a conventional MCHS. Todays demands on electronic components to become smaller and more efficient at the same time place equally high demands on the corresponding cooling devices. The use of porous material along the flow channels can enhance the heat transfer, by increasing the heat transfer surface area. At the same time, the pressure drop is also increased, requiring more pumping power. With a parametric study over the thickness of the porous substrate an optimized design of the porous MCHS can be found.

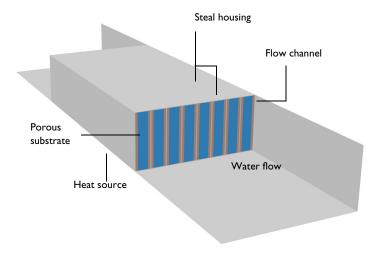


Figure 1: Geometry and operating conditions of the porous MCHS.

The design and operating conditions are taken from Ref. 1 and are illustrated in Figure 1. The problem can be reduced to modeling only one half of a single channel. This is sufficient, because the performance is mainly influenced by the pressure drop and heat transfer from the bottom boundary to the water in the channel. The geometry of the modeled domain is shown in Figure 2.

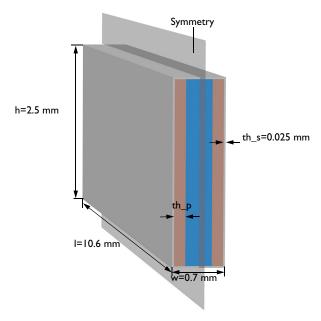


Figure 2: Geometry of the modeling domain. The free flow domain used to provide the inflow profile for the MCHS is not shown.

The flow channels contain sintered porous metal blocks with a porosity of $\epsilon=0.402$ on each side. A heat source with $q_{\rm in}=100~{\rm W/cm}^2$ is attached to the bottom. Water with an inlet velocity of $u=0.2{\rm m/s}$ and a temperature of $T_{\rm in}=300~{\rm K}$ is used as cooling fluid. The flow is assumed to be laminar, incompressible and stationary. The flow properties are also independent of the temperature field. Inside the porous domains the governing equation is the Brinkman equation with a Forchheimer correction term (also known as the Brinkman-Forchheimer or Darcy-Brinkman-Forchheimer equation). The pressure drop depends on the velocity field ${\bf u}$ as

$$-\nabla p = \frac{\mu}{\kappa} \mathbf{u} + \frac{c_F}{\sqrt{\kappa}} \rho \mathbf{u} |\mathbf{u}| \tag{1}$$

where μ (Pa·s) is the fluid viscosity, ρ (kg/m³) the density, and κ (m²) the permeability of the porous substrate.

To evaluate the performance of the porous MCHS over the conventional MCHS, the first computation solves the model assuming only free flow. Then, a second study performs a parametric sweep over the thickness of the porous substrate (th_p). The following performance parameters are evaluated:

- Pressure drop, that is the pressure difference between inlet and outlet of the porous MCHS
- 2 Average heat transfer coefficient of the MCHS, given by

$$h_{\text{mchs}} = \frac{q_{\text{in}}}{\overline{T_{\text{w}}} - T_{\text{in}}}$$
 (2)

with the average wall temperature at the bottom centerline $\overline{T_{\mathrm{w}}}$.

3 Reynolds number is defined as

$$Re = \frac{\rho u_{in} D_{h}}{\mu}$$
 (3)

with the hydraulic diameter D_h (m) that is defined based on the length and width of the free flow channel, l_f and w_f respectively, as follows:

$$D_{\rm h} = \frac{2l_{\rm f}w_{\rm f}}{l_{\rm f} + w_{\rm f}}$$

4 The Nusselt number describes the ratio of convective to conductive heat transfer according to

$$Nu = \frac{D_h h_{mchs}}{k_{\epsilon}}$$
 (4)

where $k_{\rm f}$ is the fluids thermal conductivity.

5 The Figure of Merit (FOM) compares the performance of two different designs with the following expression:

$$FOM = \frac{h_{\text{mchs}}/h_{\text{mchs, base}}}{(\Omega/\Omega_{\text{base}})^{1/3}}$$
 (5)

The index base refers to the values for the MCHS without the porous structure and $\Omega = u_{\rm in} l_{\rm f} w_{\rm f} \Delta p$ is the pumping power.

Equation 1 is valid for $1 \le \text{Re} \le 1000$. An estimation of the Reynolds number (Equation 3) results in Re ~ 300 such that the choice of the Brinkman-Forchheimer equation is valid.

Results and Discussion

Figure 3 shows the velocity field in a cross section of the channel. The velocity magnitude inside the porous structure is small (dark blue) compared to that of the free flow channel.

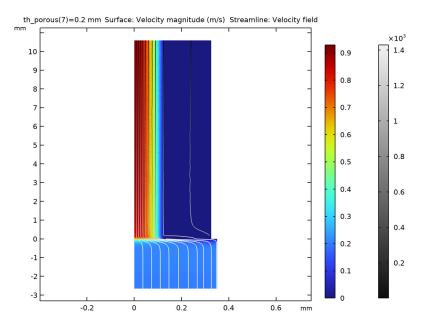


Figure 3: Cross section of the velocity field along the channel (scaled view). The dark blue color indicates the porous structure, because the velocity magnitude is small in this area.

The temperature distribution is shown together with the velocity profile in Figure 4.

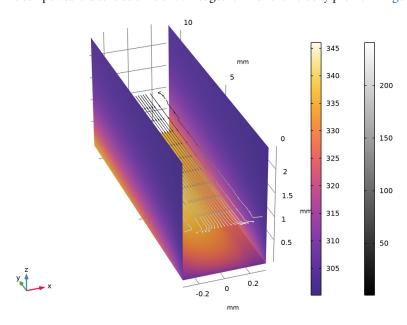


Figure 4: Temperature distribution (color) and velocity field (arrows) with the gray scale indicating the pressure.

The pressure drop and average heat transfer coefficient as a function of the thickness of the porous structure are shown in Figure 5. With increasing thickness, both values also increase.

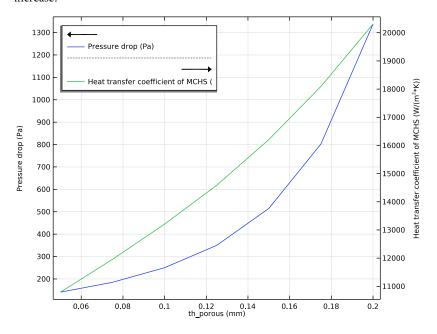


Figure 5: Pressure drop and average heat transfer coefficient.

Figure 6 shows how the dimensionless Reynolds and Nusselt numbers depend on this thickness. The Reynolds number decreases with increasing $th_{\rm p}$ and varies in the range from

100 to 210, meaning that the choice of the Brinkman–Forchheimer equation is justified. The Nusselt number has a maximum at th_p = 0.125 mm.

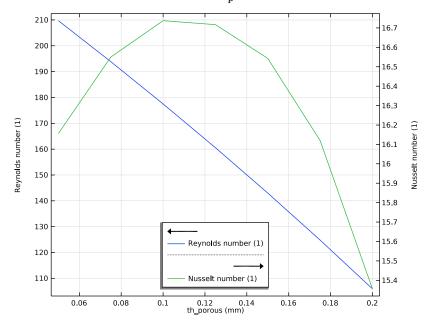


Figure 6: Reynolds and Nusselt numbers.

The Figure of Merit (Figure 7) shows that the optimal performance is achieved for $th_{p} = 0.1 \text{ mm}.$

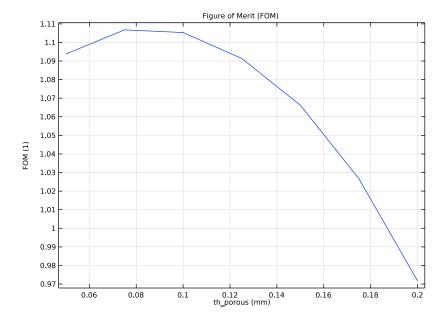


Figure 7: The Figure of Merit comparing the performances of the porous and the conventional MCHS.

Notes About the COMSOL Implementation

This model shows how to analyze the performance of the porous MCHS for varying porous substrate thickness. The model geometry and operating conditions are fully parameterized, such that you can easily extend the model for various parameters, as for example the inlet velocity or other channel dimensions.

Reference

1. A. Ghahremannezhad and K. Vafai, "Thermal and hydraulic performance enhancement of microchannel heat sinks utilizing porous substrates," Int. J. Heat Mass Transf., vol. 122, pp. 1313-1326, 2018.

Application Library path: Porous Media Flow Module/Heat Transfer/ porous_microchannel_heat_sink

Modeling Instructions

From the File menu, choose New.

NEW

In the New window, click Model Wizard.

MODEL WIZARD

- I In the Model Wizard window, click **3D**.
- 2 In the Select Physics tree, select Heat Transfer>Conjugate Heat Transfer>Laminar Flow.
- 3 Click Add.
- 4 Click Study.
- 5 In the Select Study tree, select Preset Studies for Selected Multiphysics>Stationary, One-Way NITF.
- 6 Click **Done**.

GEOMETRY I

- I In the Geometry toolbar, click Insert Sequence and choose Insert Sequence.
- 2 Browse to the model's Application Libraries folder and double-click the file porous_microchannel_heat_sink_geom_sequence.mph.
- 3 In the Geometry toolbar, click **Build All**.

GLOBAL DEFINITIONS

Parameters 1

The geometry parameters are already present after loading the file. Add a few more parameters for the material properties and operating conditions.

- I In the Model Builder window, under Global Definitions click Parameters I.
- 2 In the Settings window for Parameters, locate the Parameters section.

3 In the table, enter the following settings:

Name	Expression	Value	Description
width_channel	width-2* (th_porous+ th_solid)	4.5E-4 m	Channel width
height_channel	height-2* th_solid	0.00245 m	Channel height
rho_f	998[kg/m^3]	998 kg/m³	Density, fluid
mu_f	8.55e-4[Pa*s]	8.55E-4 Pa·s	Viscosity, fluid
k_f	0.6[W/(m*K)]	0.6 W/(m·K)	Thermal conductivity, fluid
Cp_f	4182[J/(kg*K)]	4182 J/(kg·K)	Heat capacity, fluid
por	0.404	0.404	Porosity
d_p	20[um]	2E-5 m	Pore size
карра	d_p^2/150*por^3/ (1-por)^2	4.9502E-13 m ²	Permeability
q_in	50[W/cm^2]	5E5 W/m ²	Heat load
T_in	300[K]	300 K	Inlet temperature
u_in	0.2[m/s]	0.2 m/s	Inlet velocity

Next, add the materials. For the fluid use a user-defined material with the parameters defined above. Load steel from the Material Library.

Water

- I In the Model Builder window, under Global Definitions right-click Materials and choose Blank Material.
- 2 In the Settings window for Material, type Water in the Label text field.

ADD MATERIAL

- I In the Home toolbar, click **‡ Add Material** to open the **Add Material** window.
- 2 Go to the Add Material window.
- 3 In the tree, select Built-in>Steel AISI 4340.
- 4 Click Add to Global Materials in the window toolbar.
- 5 In the Home toolbar, click 44 Add Material to close the Add Material window.

MATERIALS

Material Link I (matlnk I)

In the Model Builder window, under Component I (compl) right-click Materials and choose More Materials>Material Link.

Material Link 2 (matlnk2)

- I Right-click Materials and choose More Materials>Material Link.
- 2 In the Settings window for Material Link, locate the Link Settings section.
- 3 From the Material list, choose Steel AISI 4340 (mat2).
- 4 Locate the Geometric Entity Selection section. From the Selection list, choose Solid.

Porous Material I (pmat I)

- I Right-click Materials and choose More Materials>Porous Material.
- 2 In the Settings window for Porous Material, locate the Geometric Entity Selection section.
- 3 From the Selection list, choose Porous.

Now, set up the domain conditions. This determines which material properties are required and you can fill in the missing materials afterward. For this step the selections are helpful.

HEAT TRANSFER IN SOLIDS AND FLUIDS (HT)

Porous Medium I

- I In the Model Builder window, under Component I (compl) right-click Heat Transfer in Solids and Fluids (ht) and choose Specific Media>Porous Medium.
- 2 In the Settings window for Porous Medium, locate the Domain Selection section.
- 3 From the Selection list, choose Porous.

Porous Matrix I

- I In the Model Builder window, click Porous Matrix I.
- 2 In the Settings window for Porous Matrix, locate the Matrix Properties section.
- 3 From the Define list, choose Solid phase properties.

LAMINAR FLOW (SPF)

Assume incompressible flow.

- I In the Model Builder window, under Component I (compl) click Laminar Flow (spf).
- 2 In the Settings window for Laminar Flow, locate the Physical Model section.

- 3 From the Compressibility list, choose Incompressible flow.
- 4 Select the Enable porous media domains check box.
- 5 Locate the Domain Selection section. From the Selection list, choose Flow Domain.

Porous Medium I

- I In the Physics toolbar, click **Domains** and choose Porous Medium.
- 2 In the Settings window for Porous Medium, locate the Domain Selection section.
- 3 From the Selection list, choose Porous.
- **4** Locate the **Porous Medium** section. From the **Flow model** list, choose **Non-Darcian flow**. This enables the Forchheimer pressure drop.

Porous Matrix I

- I In the Model Builder window, click Porous Matrix I.
- 2 In the Settings window for Porous Matrix, locate the Matrix Properties section.
- **3** From the κ list, choose **User defined**. In the associated text field, type kappa.

HEAT TRANSFER IN SOLIDS AND FLUIDS (HT)

Fluid 1

- I In the Model Builder window, under Component I (compl)>
 Heat Transfer in Solids and Fluids (ht) click Fluid I.
- 2 In the Settings window for Fluid, locate the Domain Selection section.
- 3 From the Selection list, choose Flow Domain.

You can now specify the values of the missing material properties.

MATERIALS

Porous Material I (pmat I)

- I In the Model Builder window, expand the Component I (compl)>Materials> Porous Material I (pmatl) node, then click Porous Material I (pmatl).
- 2 In the Settings window for Porous Material, locate the Phase-Specific Properties section.
- 3 Click Add Required Phase Nodes.

Solid I (bmat1.solid1)

- I In the Model Builder window, click Solid I (pmat1.solid1).
- 2 In the Settings window for Solid, locate the Solid Properties section.
- 3 From the Material list, choose Steel AISI 4340 (mat2).

4 In the θ_s text field, type 1-por.

Porous Material I (pmat I)

- I In the Model Builder window, click Porous Material I (pmatl).
- 2 In the Settings window for Porous Material, locate the Homogenized Material section.
- 3 From the Material list, choose Water (matl).

This **Homogenized Material** is used for all physics features that are not related to a porous medium feature in any of the physics interfaces.

GLOBAL DEFINITIONS

Water (mat I)

- I In the Model Builder window, under Global Definitions>Materials click Water (mat1).
- 2 In the Settings window for Material, locate the Material Contents section.
- **3** In the table, enter the following settings:

Property	Variable	Value	Unit	Property group
Heat capacity at constant pressure	Ср	Cp_f	J/(kg·K)	Basic
Density	rho	rho_f	kg/m³	Basic
Thermal conductivity	k_iso ; kii = k_iso, kij = 0	k_f	W/(m·K)	Basic
Dynamic viscosity	mu	mu_f	Pa·s	Basic

Complete the physics setup by adding the boundary conditions.

HEAT TRANSFER IN SOLIDS AND FLUIDS (HT)

Boundary Heat Source I

- I In the Physics toolbar, click **Boundaries** and choose **Boundary Heat Source**.
- 2 Select Boundary 7 only.
- 3 In the Settings window for Boundary Heat Source, locate the Boundary Heat Source section.
- **4** In the Q_b text field, type q_in.

Inflow I

- I In the Physics toolbar, click **Boundaries** and choose Inflow.
- 2 In the Settings window for Inflow, locate the Boundary Selection section.

- **3** From the **Selection** list, choose **Inlet**.
- **4** Locate the **Upstream Properties** section. In the T_{ustr} text field, type T_in.

Outflow I

- I In the Physics toolbar, click **Boundaries** and choose **Outflow**.
- 2 In the Settings window for Outflow, locate the Boundary Selection section.
- **3** From the **Selection** list, choose **Outlet**.

Symmetry I

- I In the Physics toolbar, click **Boundaries** and choose Symmetry.
- 2 In the Settings window for Symmetry, locate the Boundary Selection section.
- 3 From the Selection list, choose Symmetry.

LAMINAR FLOW (SPF)

In the Model Builder window, under Component I (compl) click Laminar Flow (spf).

Inlet I

- I In the Physics toolbar, click **Boundaries** and choose Inlet.
- 2 In the Settings window for Inlet, locate the Boundary Selection section.
- 3 From the Selection list, choose Inlet.
- **4** Locate the **Velocity** section. In the U_0 text field, type u_in.

Outlet I

- I In the Physics toolbar, click **Boundaries** and choose **Outlet**.
- 2 In the Settings window for Outlet, locate the Boundary Selection section.
- **3** From the **Selection** list, choose **Outlet**.

Symmetry I

- I In the Physics toolbar, click **Boundaries** and choose Symmetry.
- 2 In the Settings window for Symmetry, locate the Boundary Selection section.
- **3** From the **Selection** list, choose **Symmetry**.

To evaluate the performance, define new variables (see Equation 2 to Equation 5).

To evaluate the pressure drop define a nonlocal average coupling at the inlet of the porous MCHS. Use the average temperature of the centerline at the bottom surface to evaluate the heat transfer coefficient.

DEFINITIONS (COMPI)

Average: Inlet of Porous MCHS

- I In the Definitions toolbar, click / Nonlocal Couplings and choose Average.
- 2 In the Settings window for Average, locate the Source Selection section.
- 3 From the Geometric entity level list, choose Boundary.
- 4 Click the Wireframe Rendering button in the Graphics toolbar.
- **5** Select Boundaries 9 and 17 only.
- 6 In the Label text field, type Average: Inlet of Porous MCHS.

Average: Centerline, Bottom Surface

- I In the Definitions toolbar, click Monlocal Couplings and choose Average.
- 2 In the Settings window for Average, locate the Source Selection section.
- 3 From the Geometric entity level list, choose Edge.
- **4** Select Edge 7 only.
- 5 In the Label text field, type Average: Centerline, Bottom Surface.

Variables 1

- I In the Model Builder window, right-click Definitions and choose Variables.
- 2 In the Settings window for Variables, locate the Variables section.
- **3** In the table, enter the following settings:

Name	Expression	Unit	Description
Dh	2*width_channel* height_channel/ (width_channel+ height_channel)	m	Hydraulic diameter
dp	aveop1(p)	Pa	Pressure drop
Tw	aveop2(T)	K	Average wall temperature
Omega	<pre>spf.out1.Mflow/ rho_f*dp</pre>	W	Pumping power
h_mchs	q_in/(Tw-T_in)	W/(m²·K)	Heat transfer coefficient of MCHS
Nu	h_mchs*Dh/k_f		Nusselt number
Re	rho_f*u_in*Dh/mu_f		Reynolds number

HEAT TRANSFER IN SOLIDS AND FLUIDS (HT)

In order to solve the model efficiently and achieve high accuracy, two aspects must be considered. The geometry has a high aspect ratio which can lead to an unnecessarily large number of mesh elements. The boundary heat source at the bottom surface leads to a high temperature gradient in z direction close to this boundary. To improve the accuracy of the heat transfer computations, use quadratic elements for the temperature discretization. Modify the physics-controlled mesh sequence to build an efficient mesh for this model.

- I In the Model Builder window, under Component I (compl) click Heat Transfer in Solids and Fluids (ht).
- 2 In the Settings window for Heat Transfer in Solids and Fluids, click to expand the Discretization section.
- 3 From the Temperature list, choose Quadratic Lagrange.

MESH I

- I In the Model Builder window, under Component I (compl) click Mesh I.
- 2 In the Settings window for Mesh, locate the Physics-Controlled Mesh section.
- 3 From the Element size list, choose Finer.
- 4 In the table, clear the Use check boxes for Heat Transfer in Solids and Fluids (ht) and Nonisothermal Flow I (nitfl).

Size

Right-click Component I (compl)>Mesh I and choose Edit Physics-Induced Sequence.

Size 1

- I In the Settings window for Size, locate the Element Size section.
- 2 Click the **Custom** button.
- 3 Locate the Element Size Parameters section.
- 4 Select the Maximum element size check box. In the associated text field, type 0.05.

Corner Refinement I

In the **Model Builder** window, right-click **Corner Refinement 1** and choose **Disable**, because it does not improve the overall accuracy. It only produces small mesh elements close to the solid domain at the inlet of the heat sink.

Free Tetrahedral I

- I In the Model Builder window, click Free Tetrahedral I.
- 2 Select Domains 3 and 4 only.

- 3 In the Settings window for Free Tetrahedral, click to expand the Scale Geometry section.
- 4 In the y-direction scale text field, type 0.5.
- 5 Click to expand the Element Quality Optimization section. From the Optimization level list, choose Medium.
- 6 Select the Avoid too small elements check box.

Size 1

- I Right-click Free Tetrahedral I and choose Size.
- 2 In the Settings window for Size, locate the Geometric Entity Selection section.
- 3 Click Clear Selection.
- 4 Select Domain 3 only.
- 5 Locate the Element Size section. From the Calibrate for list, choose Fluid dynamics.
- **6** From the **Predefined** list, choose **Fine**.
- 7 Click the **Custom** button.
- 8 Locate the Element Size Parameters section.
- 9 Select the Maximum element size check box. In the associated text field, type min(0.33* width channel/2,0.05[mm]).

This expression ensures that the resolution of the flow domain is good enough for all parameters for the thickness of the porous material.

10 Click Build Selected.

Now we can use a swept mesh for the solid domain. This is possible because the geometry contains so-called mesh control faces.

Swept I

- I In the Mesh toolbar, click A Swept.
- 2 In the Settings window for Swept, locate the Domain Selection section.
- 3 From the Geometric entity level list, choose Domain.
- 4 From the Selection list, choose Solid.

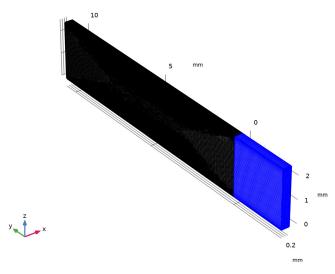
Distribution I

- I Right-click Swept I and choose Distribution.
- 2 In the Settings window for Distribution, locate the Distribution section.
- 3 In the Number of elements text field, type 2.

Swebt 2

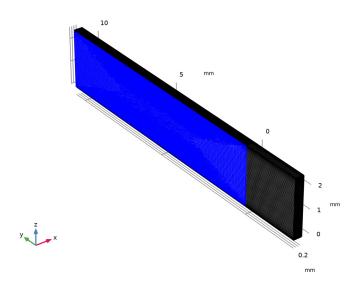
I In the Mesh toolbar, click A Swept.

2 In the Settings window for Swept, click Build Selected.



Boundary Layers 1

- I In the Model Builder window, click Boundary Layers I.
- 2 Select Domains 3 and 4 only.
- 3 In the Settings window for Boundary Layers, click Build All.



LAMINAR FLOW (SPF)

To compare the performance of the porous MCHS with that of a conventional one, ignore the porous domain in the first study. To do so, deactivate the relevant features.

Porous Medium I

In the Model Builder window, under Component I (compl)>Laminar Flow (spf) right-click Porous Medium I and choose Disable in All Studies.

HEAT TRANSFER IN SOLIDS AND FLUIDS (HT)

Porous Medium I

In the Model Builder window, under Component I (compl)> Heat Transfer in Solids and Fluids (ht) right-click Porous Medium I and choose Disable in All Studies.

STUDY I: REFERENCE MCHS

- I In the Model Builder window, click Study I.
- 2 In the Settings window for Study, type Study 1: Reference MCHS in the Label text field.
- 3 In the Home toolbar, click **Compute**.

RESULTS

Global Evaluation 1

Next, evaluate the performance parameters of the reference MCHS.

- I In the Results toolbar, click (8.5) Global Evaluation.
- 2 In the Settings window for Global Evaluation, locate the Expressions section.
- **3** In the table, enter the following settings:

Expression	Unit	Description
dp	Pa	Pressure drop
Omega	W	Pumping power
h_mchs	W/(m^2*K)	Heat transfer coefficient of MCHS
Nu	1	Nusselt number
Re	1	Reynolds number

4 Click **= Evaluate**.

ADD STUDY

Add a second study and perform a parametric sweep over the thickness of the porous substrate. Of course you can run a parametric sweep over many parameters. For this demo model a single parameter is sufficient to demonstrate the principal approach.

- I In the Home toolbar, click Add Study to open the Add Study window.
- 2 Go to the Add Study window.
- 3 Find the Studies subsection. In the Select Study tree, select Preset Studies for Selected Multiphysics>Stationary, One-Way NITF.
- 4 Click Add Study in the window toolbar.
- 5 In the Home toolbar, click Add Study to close the Add Study window.

STUDY 2: PARAMETRIC

- I In the Model Builder window, click Study 2.
- 2 In the Settings window for Study, type Study 2: Parametric in the Label text field.

Parametric Sweep

- I In the Study toolbar, click Parametric Sweep.
- 2 In the Settings window for Parametric Sweep, locate the Study Settings section.
- 3 Click + Add.
- 4 In the table, enter the following settings:

Parameter name	Parameter value list	Parameter unit
th_porous (Porous structure thickness)	range(0.05,0.025,0.2)	mm

5 In the Study toolbar, click **Compute**.

Create a cross-section plot of the velocity (Figure 3) as follows:

RESULTS

Cut Plane 1

- I In the Results toolbar, click Cut Plane.
- 2 In the Settings window for Cut Plane, locate the Data section.
- 3 From the Dataset list, choose Study 2: Parametric/Parametric Solutions I (sol5).
- 4 Locate the Plane Data section. From the Plane list, choose xy-planes.
- 5 In the z-coordinate text field, type height/2.

Velocity, Cross Section

- I In the **Results** toolbar, click 2D Plot Group.
- 2 In the Settings window for 2D Plot Group, type Velocity, Cross Section in the Label text field.
- 3 Locate the Data section. From the Dataset list, choose Cut Plane 1.
- 4 Locate the Plot Settings section. Clear the Plot dataset edges check box.

Surface I

- I Right-click Velocity, Cross Section and choose Surface.
- 2 In the Settings window for Surface, locate the Expression section.
- 3 In the Expression text field, type spf.U.

Streamline 1

- I In the Model Builder window, right-click Velocity, Cross Section and choose Streamline.
- 2 In the Settings window for Streamline, click Replace Expression in the upper-right corner of the Expression section. From the menu, choose Component I (compl)>Laminar Flow> Velocity and pressure>u,v,w - Velocity field.
- 3 Locate the Streamline Positioning section. From the Positioning list, choose Magnitude controlled.
- 4 In the Minimum distance text field, type 0.001.
- 5 In the Maximum distance text field, type 0.01.

Color Expression 1

- I Right-click Streamline I and choose Color Expression.
- 2 In the Settings window for Color Expression, locate the Coloring and Style section.
- 3 Click Change Color Table.
- 4 In the Color Table dialog box, select Linear>GrayPrint in the tree.
- 5 Click OK.
- 6 In the Settings window for Color Expression, locate the Expression section.
- 7 In the Expression text field, type p.
- 8 In the Velocity, Cross Section toolbar, click **Plot**.
- 9 In the Model Builder window, expand the Results>Views node.

Axis

- I In the Model Builder window, expand the Results>Views>View 2D 6 node, then click Axis.
- 2 In the Settings window for Axis, locate the Axis section.

- 3 From the View scale list, choose Automatic.
- 4 From the Automatic list, choose Anisotropic.
- 5 In the y weight text field, type 3.
- 6 Click (Update.
- 7 Click the Zoom Extents button in the Graphics toolbar.

Velocity, Cross Section

- I In the Model Builder window, under Results click Velocity, Cross Section.
- 2 In the Velocity, Cross Section toolbar, click Plot.

Global Evaluation 1

To analyze the performance of the porous MCHS, duplicate the **Global Evaluation 1** node and apply the new dataset.

Global Evaluation 2

- I In the Model Builder window, under Results>Derived Values right-click Global Evaluation I and choose Duplicate.
- 2 In the Settings window for Global Evaluation, locate the Data section.
- 3 From the Dataset list, choose Study 2: Parametric/Parametric Solutions I (sol5).
- 4 Click ▼ next to **= Evaluate**, then choose **New Table**.

TABLE

- I Go to the **Table** window.
- 2 Click Table Graph in the window toolbar.

RESULTS

Table Graph 1

- I In the Model Builder window, under Results>ID Plot Group 10 click Table Graph I.
- 2 In the Settings window for Table Graph, locate the Data section.
- 3 From the Plot columns list, choose Manual.
- 4 In the Columns list, select Pressure drop (Pa).
- **5** Click to expand the **Legends** section. Select the **Show legends** check box.

Table Graph 2

- I Right-click Results>ID Plot Group 10>Table Graph I and choose Duplicate.
- 2 In the Settings window for Table Graph, locate the Data section.

- 3 In the Columns list, select Heat transfer coefficient of MCHS (W/(m^2*K)).
- 4 In the ID Plot Group 10 toolbar, click Plot.

Heat-Transfer Coefficient and Pressure Drop

- I In the Model Builder window, under Results click ID Plot Group 10.
- 2 In the Settings window for ID Plot Group, type Heat-Transfer Coefficient and Pressure Drop in the Label text field.
- **3** Locate the **Plot Settings** section. Select the **Two y-axes** check box.
- 4 In the table, select the Plot on secondary y-axis check box for Table Graph 2.
- **5** Locate the **Legend** section. From the **Position** list, choose **Upper left**.
- 6 In the Heat-Transfer Coefficient and Pressure Drop toolbar, click Plot, and compare with Figure 5.

Reynolds and Nusselt Numbers

Plot the dimensionless Reynolds and Nusselt numbers in the same way.

- I Right-click Heat-Transfer Coefficient and Pressure Drop and choose Duplicate.
- 2 In the Settings window for ID Plot Group, type Reynolds and Nusselt Numbers in the Label text field.

Table Graph 1

- I In the Model Builder window, expand the Reynolds and Nusselt Numbers node, then click Table Graph I.
- 2 In the Settings window for Table Graph, locate the Data section.
- 3 In the Columns list, select Reynolds number (1).

Table Graph 2

- I In the Model Builder window, click Table Graph 2.
- 2 In the Settings window for Table Graph, locate the Data section.
- 3 In the Columns list, select Nusselt number (1).
- 4 In the Reynolds and Nusselt Numbers toolbar, click Plot, and compare with Figure 6.

Reynolds and Nusselt Numbers

- I In the Model Builder window, click Reynolds and Nusselt Numbers.
- 2 In the Settings window for ID Plot Group, locate the Legend section.
- **3** From the **Position** list, choose **Lower middle**.

Global Evaluation 3

To compare the different designs in terms of overall performance, the figure of merit can be calculated according to Equation 5 as follows:

- I In the Results toolbar, click (8.5) Global Evaluation.
- 2 In the Settings window for Global Evaluation, locate the Data section.
- 3 From the Dataset list, choose Study 2: Parametric/Parametric Solutions I (sol5).
- **4** Locate the **Expressions** section. In the table, enter the following settings:

Expression	Unit	Description
h_mchs/withsol('sol1',h_mchs)/(Omega/ withsol('sol1', Omega))^(1/3)	1	FOM

With the withsol operator, you can use results from other solutions than that of the chosen dataset.

5 Click ▼ next to **= Evaluate**, then choose **New Table**.

TABLE

- I Go to the **Table** window.
- 2 Click **Table Graph** in the window toolbar.

RESULTS

FO_M

- I In the Model Builder window, under Results click ID Plot Group 12.
- 2 In the Settings window for ID Plot Group, click to expand the Title section.
- 3 From the Title type list, choose Manual.
- 4 In the Title text area, type Figure of Merit (FOM).
- 5 In the ID Plot Group 12 toolbar, click Plot, and compare with fig Figure 7.
- 6 In the Label text field, type FOM.

With a porous substrate thickness of 0.1 mm the performance has increased by approximately 12%.

Study 2: Parametric/Parametric Solutions 1 (sol5)

To reproduce Figure 4, proceed as follows.

In the Model Builder window, under Results>Datasets click Study 2: Parametric/ Parametric Solutions I (sol5).

Selection

- I In the Results toolbar, click has a Attributes and choose Selection.
- 2 In the Settings window for Selection, locate the Geometric Entity Selection section.
- 3 From the Geometric entity level list, choose Domain.
- **4** Select Domains 3 and 4 only.

Mirror 3D I

- I In the Results toolbar, click More Datasets and choose Mirror 3D.
- 2 In the Settings window for Mirror 3D, locate the Data section.
- 3 From the Dataset list, choose Study 2: Parametric/Parametric Solutions I (sol5).

Surface 3

- I In the Results toolbar, click More Datasets and choose Surface.
- 2 In the Settings window for Surface, locate the Data section.
- 3 From the Dataset list, choose Study 2: Parametric/Parametric Solutions I (sol5).
- 4 Select Boundaries 10, 18, and 21 only.

Mirror 3D 2

- I In the Results toolbar, click More Datasets and choose Mirror 3D.
- 2 In the Settings window for Mirror 3D, locate the Data section.
- 3 From the Dataset list, choose Surface 3.

Cut Plane 2

- I In the Results toolbar, click Cut Plane.
- 2 In the Settings window for Cut Plane, locate the Data section.
- 3 From the Dataset list, choose Mirror 3D 1.
- 4 Locate the Plane Data section. From the Plane list, choose xy-planes.
- 5 In the z-coordinate text field, type 1.

Velocity and Temperature Fields

- I In the Results toolbar, click 3D Plot Group.
- 2 In the Settings window for 3D Plot Group, type Velocity and Temperature Fields in the Label text field.
- 3 Locate the Data section. From the Dataset list, choose Study 2: Parametric/ Parametric Solutions I (sol5).
- 4 From the Parameter value (th porous (mm)) list, choose 0.1.

Surface I

- I Right-click Velocity and Temperature Fields and choose Surface.
- 2 In the Settings window for Surface, locate the Data section.
- 3 From the Dataset list, choose Mirror 3D 2.
- 4 From the Solution parameters list, choose From parent.
- 5 Locate the Coloring and Style section. Click Change Color Table.
- 6 In the Color Table dialog box, select Thermal>HeatCameraLight in the tree.
- 7 Click OK.

Velocity and Temperature Fields

In the Model Builder window, click Velocity and Temperature Fields.

Streamline Surface 1

- I In the Velocity and Temperature Fields toolbar, click More Plots and choose Streamline Surface.
- 2 In the Settings window for Streamline Surface, locate the Data section.
- 3 From the Dataset list, choose Cut Plane 2.
- 4 Click Replace Expression in the upper-right corner of the Expression section. From the menu, choose Component I (compl)>Laminar Flow>Velocity and pressure>u,v,w -Velocity field.
- 5 Locate the Data section. From the Solution parameters list, choose From parent.
- 6 Locate the Streamline Positioning section. From the Positioning list, choose Magnitude controlled.
- 7 In the Minimum distance text field, type 0.002.
- 8 In the Maximum distance text field, type 0.01.
- 9 Locate the Coloring and Style section. Find the Point style subsection. From the Type list, choose Arrow.
- 10 From the Arrow type list, choose Cone.

Color Expression 1

- I Right-click Streamline Surface I and choose Color Expression.
- 2 In the Settings window for Color Expression, locate the Expression section.
- **3** In the **Expression** text field, type p.
- 4 Locate the Coloring and Style section. Click Change Color Table.
- 5 In the Color Table dialog box, select Linear>GrayScale in the tree.

6 Click OK.

Velocity and Temperature Fields

- I In the Model Builder window, under Results click Velocity and Temperature Fields.
- 2 In the Settings window for 3D Plot Group, locate the Plot Settings section.
- 3 Clear the Plot dataset edges check box.
 - Add a view to get a better impression.
- 4 Click the Show More Options button in the Model Builder toolbar.
- 5 In the Show More Options dialog box, in the tree, select the check box for the node Results>Views.
- 6 Click OK.

View 3D 7

In the Model Builder window, under Results right-click Views and choose View 3D.

Camera

- I In the Model Builder window, expand the View 3D 7 node, then click Camera.
- 2 In the Settings window for Camera, locate the Camera section.
- 3 From the View scale list, choose Manual.
- 4 In the x scale text field, type 5.
- 5 In the z scale text field, type 2.

Rotate the geometry to get a similar image as Figure 4.