

# Heat Generation in a Disc Brake

## Introduction

Cars need brakes for obvious reasons, and you do not want these to fail. Brake failure can be caused by many things, one of which is the overheating of the brake's disc. This example models the heat generation and dissipation in a disc brake of an ordinary car during panic braking and the following release period. When the driver is pressing down on the brakes, kinetic energy is transformed into thermal energy. If the brake discs overheat, temperature overload can change the material properties of the brake causing it to fade. Braking power already starts to fade at temperatures above 600 K. This is why it is so important during the design-stages to simulate the transient heating and convective cooling to figure out what the minimum interval between a series of brake engagements is.

In this application, an 1,800 kg car is traveling at 25 m/s (90 km/h or about 56 mph), until the driver suddenly panic brakes for 2 seconds. At that point the eight brake pads slow the car down at a rate of 10 m/s<sup>2</sup> (assuming the wheels do not skid against the road). Upon braking for two seconds the driver releases the brake, leaving the car traveling at 5 m/s for eight seconds without engaging the brakes. The questions to analyze with the model are:

- How hot do the brake discs and pads get when the brake is engaged?
- How much do the discs and pads cool down during the rest that follows the braking?

Model the brake disc as a 3D solid with shape and dimensions as in Figure 1. The disc has a radius of 0.14 m and a thickness of 0.013 m.

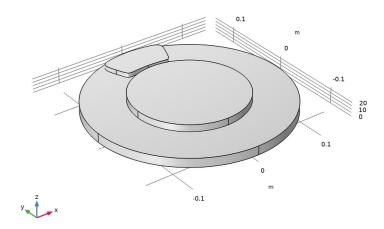


Figure 1: Model geometry, including disc and pad.

The model also includes heat conduction in the disc and the pad through the transient heat transfer equation. The heat dissipation from the disc and pad surfaces to the surrounding air is described by both convection and radiation. Table 1 summarizes the thermal properties of the materials used in this application (Ref. 1).

TABLE I: MATERIAL PROPERTIES.

PROPERTY	DESCRIPTION	DISC	PAD	AIR
$\rho$ (kg/m <sup>3</sup> )	Density	7870	2000	1.170
$C_p$ (J/(kg·K))	Heat capacity at constant pressure	449	935	1100
k (W/(m·K))	Thermal conductivity	82	8.7	0.026
ε	Surface emissivity	0.28	0.8	-

After 2 s, contact is made at the interface between the disc and the pad. Neglecting drag and other losses outside the brakes, the brakes' retardation power is given by the negative of the time derivative of the car's kinetic energy:

$$P = -\frac{d}{dt} \left( \frac{mv^2}{2} \right) = -mv \frac{dv}{dt}$$

Here m is the car's mass (1800 kg) and v denotes its speed. Figure 2 shows the profile of v and Figure 3 shows the corresponding acceleration profile.

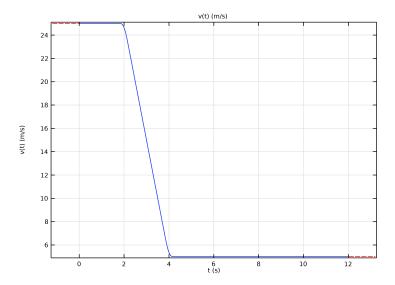


Figure 2: Velocity profile of the disc.

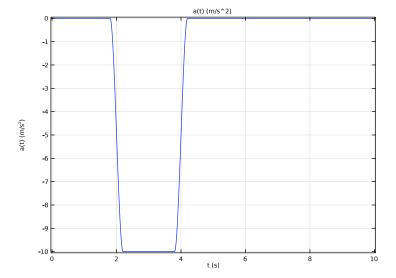


Figure 3: Acceleration profile of the disc.

At one of the eight brakes, the frictional heat source is:

$$P_{\rm b} = \frac{P}{8} = -\frac{1}{8} m v \frac{dv}{dt}$$

The contact pressure between the disc and the pad is related to the frictional heat source per unit area,  $Q_b=P_b/A$ , according to:

$$p = \frac{Q_{\rm b}}{\mu v}$$

where the friction coefficient  $\mu$  is here equal to 0.3.

The disc and pad dissipate the heat produced at the boundary between the brake pad and the disc by convection and radiation. This example models the rotation as convection in the disc. The local disc velocity vector is

$$\mathbf{v}_{\mathrm{d}} = \frac{v}{R}(-y, x)$$

At the end of the computation, produced and dissipated heat can be recovered using the relations

$$\begin{split} W_{\text{prod}} &= \int_{0}^{t_{0}} Q_{\text{prod}} dt \\ W_{\text{diss}} &= \int_{0}^{t_{0}} Q_{\text{diss}} dt \end{split} \tag{1}$$

# Results and Discussion

The surface temperatures of the disc and the pad vary with both time and position. At the contact surface between the pad and the disc the temperature increases when the brake is engaged and then decreases again as the brake is released. You can best see these results in COMSOL Multiphysics by generating an animation. Figure 4 displays the surface temperatures just before the end of the braking. A "hotspot" is visible at the contact between the brake pad and disc, just at the pad's edge. This is the area that could overheat to the point of brake failure or fade. The figure also shows the temperature decreasing along the rotational trace after the pad. During the rest, the temperature becomes significantly lower and more uniform in the disc and the pad.

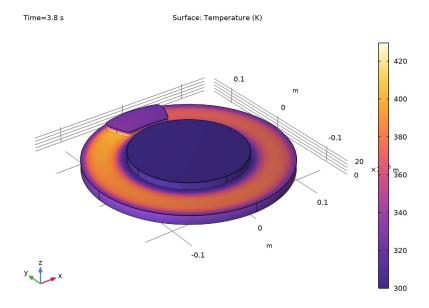


Figure 4: Surface temperature of the brake disc and pad just before releasing the brake (t = 3.8 s).

To investigate the position of the hotspot and the time of the temperature maximum, it is helpful to plot temperature versus time along the line from the center to the pad's edge shown in Figure 5. The result is displayed in Figure 6. You can see that the maximum temperature is approximately 430 K. The hotspot is positioned close to the radially outer edge of the pad. The highest temperature occurs approximately 1 s after engaging the brake.

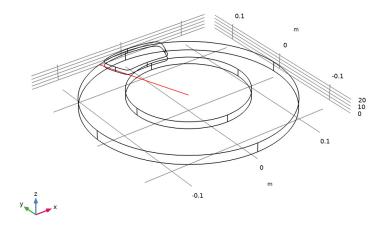


Figure 5: The radial line probed in the temperature vs. time plot in Figure 6.

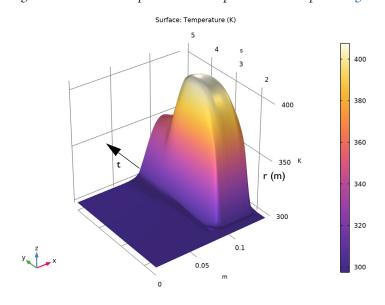


Figure 6: Temperature profile along the line indicated in Figure 5 at the disc surface (z = 0.013 m) as a function of time.

To investigate how much of the generated heat is dissipated to the air, study the surface integrals of the produced heat and the dissipated heat. These integrals give the total heat rate (W) for heat production,  $Q_{\text{prod}}$ , and heat dissipation,  $Q_{\text{diss}}$ , as functions of time for the brake disc. The time integrals of these two quantities,  $W_{\text{prod}}$  and  $W_{\text{diss}}$ , give the total heat (J) produced and dissipated, respectively, in the brake disc. Figure 7 shows a plot of the total produced heat and dissipated heat versus time. Eight seconds after the driver has stopped braking, a mere fraction of the produced heat has been dissipated. In other words, in order to cool down the system sufficiently the brake needs to remain disengaged for a much longer period than these eight seconds (100 seconds, in fact).

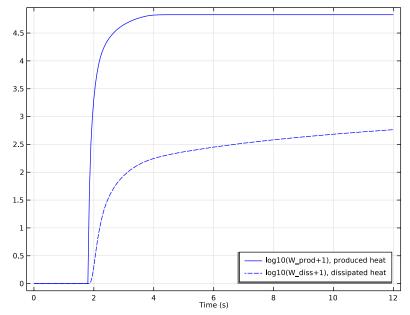


Figure 7: Comparison of total produced heat (solid line) and dissipated heat (dashed).

The results of this application can help engineers investigate how much abuse, in terms of specific braking sequences, a certain brake-disc design can tolerate before overheating. It is also possible to modify the parameters affecting the heat dissipation and investigate their influence.

# Reference

1. J.M. Coulson and J.F. Richardson, Chemical Engineering, vol. 1, eq. 9.88; material properties from appendix A2.

# Application Library path: Heat Transfer Module/

Thermal\_Contact\_and\_Friction/brake\_disc

# Modeling Instructions

From the File menu, choose New.

#### NEW

In the New window, click Model Wizard.

#### MODEL WIZARD

- I In the Model Wizard window, click **3D**.
- 2 In the Select Physics tree, select Heat Transfer>Heat Transfer in Solids (ht).
- 3 Click Add.
- 4 Click Study.
- 5 In the Select Study tree, select General Studies>Time Dependent.
- 6 Click Mone.

#### **GEOMETRY I**

Define the global parameters by loading the corresponding text file provided.

#### **GLOBAL DEFINITIONS**

Parameters 1

- I In the Model Builder window, under Global Definitions click Parameters I.
- 2 In the Settings window for Parameters, locate the Parameters section.
- 3 Click Load from File.
- 4 Browse to the model's Application Libraries folder and double-click the file brake disc parameters.txt.

#### **GEOMETRY I**

Cylinder I (cyl1)

- I In the Geometry toolbar, click ( Cylinder.
- 2 In the Settings window for Cylinder, locate the Size and Shape section.

- 3 In the Radius text field, type 0.14.
- 4 In the Height text field, type 0.013.
- 5 In the Geometry toolbar, click **Build All**.

# Cylinder 2 (cyl2)

- I In the Geometry toolbar, click ( Cylinder.
- 2 In the Settings window for Cylinder, locate the Size and Shape section.
- 3 In the Radius text field, type 0.08.
- 4 In the Height text field, type 0.01.
- **5** Locate the **Position** section. In the **z** text field, type 0.013.
- 6 In the Geometry toolbar, click **Build All**.

## Work Plane I (wbl)

- I In the Geometry toolbar, click Work Plane.
- 2 In the Settings window for Work Plane, locate the Plane Definition section.
- 3 In the z-coordinate text field, type 0.013.
- 4 Click Show Work Plane.

Work Plane I (wpl)>Cubic Bézier I (cbl)

- I In the Work Plane toolbar, click More Primitives and choose Cubic Bézier.
- 2 In the Settings window for Cubic Bézier, locate the Control Points section.
- 3 In row 1, set yw to 0.135.
- 4 In row 2, set xw to 0.02, and yw to 0.135.
- 5 In row 3, set xw to 0.05, and yw to 0.13.
- 6 In row 4, set xw to 0.04, and yw to 0.105.
- 7 Locate the Weights section. In the 3 text field, type 2.5.

Work Plane I (wbl)>Cubic Bézier 2 (cb2)

- I In the Work Plane toolbar, click \* More Primitives and choose Cubic Bézier.
- 2 In the Settings window for Cubic Bézier, locate the Control Points section.
- 3 In row 1, set xw to 0.04.
- 4 In row I, set yw to 0.105, and yw to 0.03., and yw to 0.08.
- 5 In row 3, set xw to 0.035, and yw to 0.09., and yw to 0.09.

Work Plane I (wp I)>Cubic Bézier 3 (cb3)

I In the Work Plane toolbar, click More Primitives and choose Cubic Bézier.

- 2 In the Settings window for Cubic Bézier, locate the Control Points section.
- 3 In row 1, set yw to 0.09.
- 4 In row 2, set xw to -0.035, and yw to 0.09.
- 5 In row 3, set xw to -0.03, and yw to 0.08.
- 6 In row 4, set xw to -0.04, and yw to 0.105.

Work Plane I (wp I)>Cubic Bézier 4 (cb4)

- I In the Work Plane toolbar, click More Primitives and choose Cubic Bézier.
- 2 In the Settings window for Cubic Bézier, locate the Control Points section.
- 3 In row 1, set xw to -0.04.
- 4 In row 2, set xw to -0.05, and yw to 0.09., and yw to 0.13.
- **5** In row **3**, set **xw** to -0.02, and **yw** to 0.135.
- **6** In row **4**, set **yw** to **0.135**.
- 7 Locate the Weights section. In the 2 text field, type 2.5.
- 8 In the Work Plane toolbar, click | Build All.

Work Plane I (wbl)>Convert to Solid I (csoll)

- I In the Work Plane toolbar, click Conversions and choose Convert to Solid.
- 2 Click in the **Graphics** window and then press Ctrl+A to select all objects.
- 3 In the Work Plane toolbar, click Build All.

Extrude | (extl)

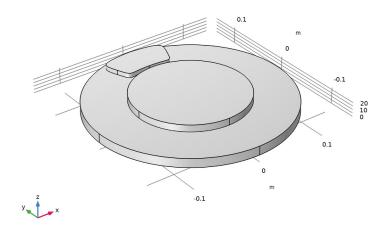
- I In the Model Builder window, under Component I (compl)>Geometry I right-click Work Plane I (wpl) and choose Extrude.
- 2 In the Settings window for Extrude, locate the Distances section.
- **3** In the table, enter the following settings:

Distances (m)

0.0065

4 In the Geometry toolbar, click Build All.

The model geometry is now complete.



Next, define some selections of certain boundaries. You will use them when defining the settings for component couplings, boundary conditions, and so on.

#### DEFINITIONS

#### Disc Faces

- 2 In the Settings window for Explicit, type Disc Faces in the Label text field.
- 3 Locate the Input Entities section. From the Geometric entity level list, choose Boundary.
- **4** Select Boundaries 1, 2, 4–6, 8, 13–15, and 18 only.

#### Pad Faces

- I In the **Definitions** toolbar, click **\( \bigcap\_{\text{a}} \) Explicit**.
- 2 In the Settings window for Explicit, type Pad Faces in the Label text field.
- 3 Locate the Input Entities section. From the Geometric entity level list, choose Boundary.
- 4 Select Boundaries 9, 10, 12, 16, and 17 only.

## Contact Faces

I In the **Definitions** toolbar, click **\( \frac{1}{2} \) Explicit**.

- 2 In the Settings window for Explicit, type Contact Faces in the Label text field.
- 3 Locate the Input Entities section. From the Geometric entity level list, choose Boundary.
  To select the contact surface boundary, it is convenient to temporarily switch to wireframe rendering.
- 4 Click the Wireframe Rendering button in the Graphics toolbar.
- **5** Select Boundary 11 only.
- **6** Click the Wireframe Rendering button in the Graphics toolbar again to return to the original state.

## External Surfaces

- I In the **Definitions** toolbar, click **\( \bigcap\_{\bigcap} \) Explicit**.
- 2 In the Settings window for Explicit, type External Surfaces in the Label text field.
- 3 Locate the Input Entities section. Select the All domains check box.
- **4** Locate the **Output Entities** section. From the **Output entities** list, choose **Adjacent boundaries**.

These instructions make you select the external boundaries of the wheel and the pad.

# Integration I (intop I)

- I In the Definitions toolbar, click Nonlocal Couplings and choose Integration.
- 2 In the Settings window for Integration, locate the Source Selection section.
- 3 From the Geometric entity level list, choose Boundary.
- 4 From the Selection list, choose Contact Faces.

## Integration 2 (intob2)

- I In the **Definitions** toolbar, click **Nonlocal Couplings** and choose **Integration**.
- 2 In the Settings window for Integration, locate the Source Selection section.
- 3 From the Geometric entity level list, choose Boundary.
- 4 From the Selection list, choose External Surfaces.

Now, define the velocity and acceleration of the car through these two piecewise and analytic functions.

# Piecewise I (pw I)

- I In the **Definitions** toolbar, click  $\bigwedge$  **Piecewise**.
- 2 In the Settings window for Piecewise, type v in the Function name text field.
- 3 Locate the **Definition** section. In the **Argument** text field, type t.
- 4 From the Smoothing list, choose Continuous second derivative.

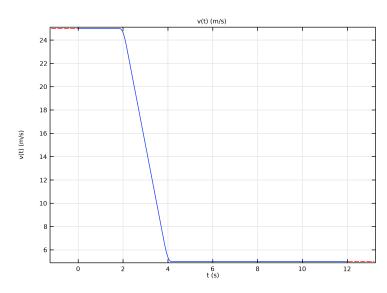
- 5 From the Transition zone list, choose Absolute size.
- 6 In the Size of transition zone text field, type 0.2.

The function definition expects nondimensional quantities for the interval bounds, and the function values. The function definition below uses unit conversions to do so.

7 Find the Intervals subsection. In the table, enter the following settings:

Start	End	Function
0	t_brake_start[1/s]	v0[s/m]
t_brake_start[1/s]	t_brake_end[1/s]	v0[s/m]+a0*(t[s]- t_brake_start)[s/m]
t_brake_end[1/s]	12	v0[s/m]+a0*(t_brake_end- t_brake_start)[s/m]

- 8 Locate the Units section. In the Arguments text field, type s.
- 9 In the Function text field, type m/s.
- 10 Click Plot.



Analytic I (an I)

- I In the **Definitions** toolbar, click  $\stackrel{f(x)}{Q}$  **Analytic**.
- 2 In the Settings window for Analytic, type a in the Function name text field.
- 3 Locate the **Definition** section. In the **Expression** text field, type d(v(t),t).
- 4 In the Arguments text field, type t.

**5** Locate the **Units** section. In the table, enter the following settings:

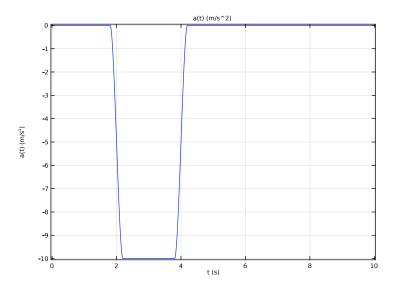
Argument	Unit
t	S

6 In the Function text field, type m/s^2.

7 Locate the Plot Parameters section. In the table, enter the following settings:

Argument	Lower limit	Upper limit	Unit
t	0	10	s

8 Click Plot.



## MATERIALS

Disc

I In the Materials toolbar, click **Blank Material**.

2 In the Settings window for Material, type Disc in the Label text field.

**3** Locate the **Material Contents** section. In the table, enter the following settings:

Property	Variable	Value	Unit	Property group
Thermal conductivity	k_iso ; kii = k_iso, kij = 0	82	W/(m·K)	Basic
Density	rho	7870	kg/m³	Basic
Heat capacity at constant pressure	Ср	449	J/(kg·K)	Basic

#### Pad

- I In the Materials toolbar, click Blank Material.
- 2 In the Settings window for Material, type Pad in the Label text field.
- **3** Select Domain 3 only.
- **4** Locate the **Material Contents** section. In the table, enter the following settings:

Property	Variable	Value	Unit	Property group
Thermal conductivity	k_iso ; kii = k_iso, kij = 0	8.7	W/(m·K)	Basic
Density	rho	2000	kg/m³	Basic
Heat capacity at constant pressure	Ср	935	J/(kg·K)	Basic

# HEAT TRANSFER IN SOLIDS (HT)

Solid with Translational Motion 1

- I In the Model Builder window, under Component I (compl) right-click Heat Transfer in Solids (ht) and choose Specific Media>Solid with Translational Motion.
- 2 In the Settings window for Solid with Translational Motion, locate the Domain Selection section.
- 3 From the Selection list, choose All domains.

Translational Motion 1

- I In the Model Builder window, click Translational Motion I.
- 2 Select Domains 1 and 2 only.
- 3 In the Settings window for Translational Motion, locate the Translational Motion section.

**4** Specify the  $\mathbf{u}_{trans}$  vector as

-y*v(t)/r_wheel	x
x*v(t)/r_wheel	у
0	z

#### Heat Flux 1

- I In the Physics toolbar, click **Boundaries** and choose **Heat Flux**.
- 2 In the Settings window for Heat Flux, locate the Boundary Selection section.
- 3 From the Selection list, choose All boundaries.
- 4 Locate the Heat Flux section. From the Flux type list, choose Convective heat flux.
- 5 From the Heat transfer coefficient list, choose External forced convection.
- 6 In the L text field, type 0.14.
- **7** In the U text field, type v(t).
- **8** In the  $T_{\text{ext}}$  text field, type T\_air.

#### Thermal Contact I

- I In the Physics toolbar, click **Boundaries** and choose **Thermal Contact**.
- **2** Select Boundary 11 only.
- 3 In the Settings window for Thermal Contact, locate the Contact Surface Properties section.
- 4 In the p text field, type ht.tc1.Qb/(mu\*v(t)).
- **5** In the  $H_c$  text field, type 800[MPa].
- **6** Locate the **Thermal Friction** section. Click the **Heat rate** button.
- 7 In the  $P_b$  text field, type -m\_car\*v(t)\*a(t)/8.

## Initial Values 1

- I In the Model Builder window, click Initial Values I.
- 2 In the Settings window for Initial Values, locate the Initial Values section.
- **3** In the *T* text field, type T\_air.

#### Surface-to-Ambient Radiation 1

- I In the Physics toolbar, click boundaries and choose Surface-to-Ambient Radiation.
- 2 In the Settings window for Surface-to-Ambient Radiation, locate the Boundary Selection section.
- 3 From the Selection list, choose Disc Faces.

- 4 Locate the Surface-to-Ambient Radiation section. From the ε list, choose User defined. In the associated text field, type 0.28.
- **5** In the  $T_{\text{amb}}$  text field, type T\_air.

Surface-to-Ambient Radiation 2

- I In the Physics toolbar, click **Boundaries** and choose Surface-to-Ambient Radiation.
- 2 In the Settings window for Surface-to-Ambient Radiation, locate the Boundary Selection section.
- 3 From the Selection list, choose Pad Faces.
- 4 Locate the Surface-to-Ambient Radiation section. From the  $\varepsilon$  list, choose User defined. In the associated text field, type 0.8.
- **5** In the  $T_{\text{amb}}$  text field, type T\_air.

Symmetry I

- I In the Physics toolbar, click **Boundaries** and choose Symmetry.
- 2 Select Boundary 3 only.

To compute the produced and dissipated heats, integrate the corresponding heat rate variables, Q prod and Q diss, over time. For this purpose, define two ODEs using a Global Equations node.

- 3 Click the Show More Options button in the Model Builder toolbar.
- 4 In the Show More Options dialog box, in the tree, select the check box for the node Physics>Equation-Based Contributions.
- 5 Click OK.

Global Equations 1

- I In the Physics toolbar, click A Global and choose Global Equations.
- 2 In the Settings window for Global Equations, locate the Units section.
- 3 Click Select Dependent Variable Quantity.
- 4 In the Physical Quantity dialog box, type energy in the text field.
- 5 Click **Filter**.
- 6 In the tree, select General>Energy (1).
- 7 Click OK.
- 8 In the Settings window for Global Equations, locate the Units section.
- 9 Click Select Source Term Quantity.
- 10 In the Physical Quantity dialog box, type power in the text field.

II Click **Filter**.

12 In the tree, select General>Power (W).

I3 Click OK.

14 In the Settings window for Global Equations, locate the Global Equations section.

**I5** In the table, enter the following settings:

Name	f(u,ut,utt,t) (W)	Initial value (u_0) (J)	Initial value (u_t0) (W)	Description
W_prod	W_prodt- intop1(ht.tc1.Qb)	0	0	Produced heat
W_diss	<pre>W_disst+ (intop2(ht.q0+ ht.rflux))</pre>	0	0	Dissipated heat

Here, W prodt (resp. W disst) is COMSOL Multiphysics syntax for the time derivative of W prod (resp. W diss). The quantities intop1(ht.tc1.Qb) and intop2(ht.q0+ ht.rflux) correspond to Q\_prod and Q\_diss. The table thus defines the first-order ODEs corresponding to Equation 1, so that W\_prod and W\_diss host the produced and dissipated heats. The initial values follow from setting t = 0.

#### MESH I

Free Triangular 1

- I In the Mesh toolbar, click A Boundary and choose Free Triangular.
- 2 Click the Transparency button in the Graphics toolbar.
- **3** Select Boundaries 4, 7, and 11 only.
- 4 Click the Transparency button in the Graphics toolbar again to return to the original state.

Size

- I In the Model Builder window, click Size.
- 2 In the Settings window for Size, locate the Element Size section.
- 3 From the Predefined list, choose Extra fine.

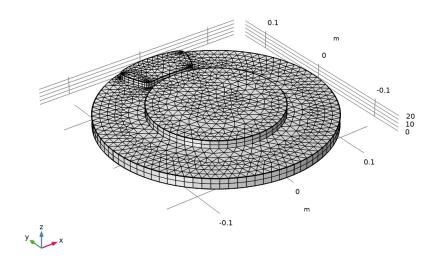
Swept I

In the Mesh toolbar, click Swept.

Distribution I

I Right-click Swept I and choose Distribution.

- 2 In the Settings window for Distribution, locate the Distribution section.
- 3 In the Number of elements text field, type 2.
- 4 In the Model Builder window, right-click Mesh I and choose Build All. The complete mesh consists of roughly 5800 elements.



## STUDY I

# Step 1: Time Dependent

- I In the Model Builder window, under Study I click Step I: Time Dependent.
- 2 In the Settings window for Time Dependent, locate the Study Settings section.
- 3 In the Output times text field, type range (0,0.5,1.5) range (1.55,0.05,3)range(3.2,0.2,5) range(6,1,12).

## Solution I (soll)

- I In the Study toolbar, click Show Default Solver.
- 2 In the Model Builder window, expand the Solution I (soll) node, then click Time-Dependent Solver 1.
- 3 In the Settings window for Time-Dependent Solver, click to expand the Time Stepping section.

- 4 From the Steps taken by solver list, choose Intermediate. This setting forces the solver to take at least one step in each specified interval.
- 5 In the Study toolbar, click **Compute**.

#### RESULTS

The first of the two default plots displays the surface temperature of the brake disc and pad at the end of the simulation interval. Modify this plot to show the time step just before releasing the brake.

#### Temperature (ht)

- I In the Model Builder window, under Results click Temperature (ht).
- 2 In the Settings window for 3D Plot Group, locate the Data section.
- 3 From the Time (s) list, choose 3.8.

# Surface 2

- I In the Model Builder window, expand the Temperature (ht) node, then click Surface 2.
- 2 In the Settings window for Surface, click to expand the Title section.
- **3** From the **Title type** list, choose **None**.

#### Surface 3

- I In the Model Builder window, click Surface 3.
- 2 In the Settings window for Surface, locate the Title section.
- 3 From the Title type list, choose None.
- 4 In the Temperature (ht) toolbar, click Plot.

Compare the result to the plot shown in Figure 4.

To compare the total produced heat and the dissipated heat, as done in Figure 7, follow the steps given below.

#### Dissipated and Produced Heats

- I In the Home toolbar, click **Add Plot Group** and choose ID Plot Group.
- 2 In the Settings window for ID Plot Group, type Dissipated and Produced Heats in the Label text field.
- **3** Click to expand the **Title** section. From the **Title type** list, choose **None**.
- **4** Click to collapse the **Title** section. Locate the **Plot Settings** section.
- 5 Select the x-axis label check box. In the associated text field, type Time (s).
- **6** Locate the **Legend** section. From the **Position** list, choose **Lower right**.

# Point Graph 1

- I In the Dissipated and Produced Heats toolbar, click Point Graph.
- 2 Select Point 1 only.
- 3 In the Settings window for Point Graph, locate the y-Axis Data section.
- **4** In the **Expression** text field, type log10(W\_prod+1).
- 5 Click to expand the Coloring and Style section. From the Color list, choose Blue.
- **6** Click to expand the **Legends** section. Select the **Show legends** check box.
- 7 From the Legends list, choose Manual.
- **8** In the table, enter the following settings:

# Legends log10(W\_prod+1), produced heat

# Point Graph 2

- I Right-click Point Graph I and choose Duplicate.
- 2 In the Settings window for Point Graph, locate the y-Axis Data section.
- **3** In the **Expression** text field, type log10(W\_diss+1).
- 4 Locate the Coloring and Style section. Find the Line style subsection. From the Line list, choose Dashed.
- **5** Locate the **Legends** section. In the table, enter the following settings:

# Legends log10(W diss+1), dissipated heat

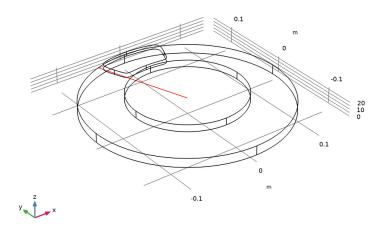
Dissipated and Produced Heats

Finally, follow the steps below to reproduce the plot in Figure 6.

#### Cut Line 3D I

- I In the Results toolbar, click Cut Line 3D.
- 2 In the Settings window for Cut Line 3D, locate the Line Data section.
- 3 In row **Point 1**, set **Z** to 0.013.
- 4 In row Point 2, set X to -0.047, y to 0.1316, and z to 0.013.

## 5 Click Plot.



## Parametric Extrusion ID I

- I In the Results toolbar, click More Datasets and choose Parametric Extrusion ID.
- 2 In the Settings window for Parametric Extrusion ID, locate the Data section.
- 3 From the Time selection list, choose From list.
- 4 Click and shift-click in the list to select all time steps from 1.5 s through 5 s.

# Temperature Profile vs. Time

- I In the Results toolbar, click 2D Plot Group.
- 2 In the Settings window for 2D Plot Group, type Temperature Profile vs. Time in the Label text field.

## Surface I

- I In the Temperature Profile vs. Time toolbar, click Surface.
- 2 In the Settings window for Surface, locate the Coloring and Style section.
- 3 Click Change Color Table.
- 4 In the Color Table dialog box, select Thermal>HeatCameraLight in the tree.
- 5 Click OK.

# Height Expression 1

I In the Temperature Profile vs. Time toolbar, click | Height Expression.

2 Click Plot.

In order to visualize the temperature on each side of the thermal contact, follow the next steps.

Temperature (ht) I

In the Model Builder window, right-click Temperature (ht) and choose Duplicate.

Surface 2

In the Model Builder window, under Results>Temperature (ht) right-click Surface 2 and choose Delete.

Surface 3

In the Model Builder window, right-click Surface 3 and choose Delete.

Surface 2

- I In the Model Builder window, expand the Results>Temperature (ht) I node, then click Surface 2.
- 2 In the Settings window for Surface, click to expand the Inherit Style section.
- **3** From the **Plot** list, choose **None**.

Surface 1

In the Model Builder window, under Results>Temperature (ht) I right-click Surface I and choose Delete.

Contact temperatures (ht)

- I In the Model Builder window, under Results click Temperature (ht) I.
- 2 In the Settings window for 3D Plot Group, type Contact temperatures (ht) in the Label text field.

Upside

- I In the Model Builder window, under Results>Contact temperatures (ht) click Surface 2.
- 2 In the Settings window for Surface, type Upside in the Label text field.
- **3** Locate the **Expression** section.
- 4 Select the **Description** check box. In the associated text field, type Upside temperature.
- 5 Locate the Coloring and Style section. Click Change Color Table.
- 6 In the Color Table dialog box, select Thermal>HeatCameraLight in the tree.
- 7 Click OK.

## Downside

- I In the Model Builder window, under Results>Contact temperatures (ht) click Surface 3.
- 2 In the Settings window for Surface, type Downside in the Label text field.
- 3 Locate the Expression section.
- 4 Select the **Description** check box. In the associated text field, type **Downside** temperature.

## Deformation

- I In the Model Builder window, expand the Upside node, then click Deformation.
- 2 In the Settings window for Deformation, locate the Scale section.
- 3 In the Scale factor text field, type 10.
- 4 In the Contact temperatures (ht) toolbar, click Plot.