

Pressure Reciprocity Calibration Coupler with Detailed Moist Air Material Properties

Introduction

When high-fidelity measurement microphones are calibrated, a pressure reciprocity calibration method is used. During calibration, two microphones are connected at each end of a closed cylindrical cavity. For the calibration procedure, it is important to understand the acoustic field inside such a cavity, including all the thermoviscous acoustic effects, for example, the acoustic boundary layers at higher frequencies and the transition to isothermal behavior at the lower frequencies.

This model sets up a simple calibration coupler model and discusses important considerations when performing a high-precision absolute-value simulation. The model results include the acoustic transfer impedance used for reciprocity calibration and the pressure in the coupler. The results are compared with analytical predictions.

The model also includes precise material property estimation using the Thermodynamics functionality in COMSOL Multiphysics. This allows setting up a moist air material that depends on the ambient pressure, temperature, and relative humidity. This functionality is available with the Liquid & Gas Properties Module. The model can also be set up using the default air material found in the Material Library, but this will represent dry air.



Figure 1: Sketch of a typical pressure reciprocity calibration coupler with two microphones. The cylindrical rz-coordinate system used in the model is also indicated, as well as the source and receiver boundaries.

Model Definition

In order to calibrate microphones a commonly used technique is the reciprocity calibration approach. The method relies on using microphones both as source and receivers (they are

reciprocal) in a well described coupled setup (see Ref. 1 and 2). The setup consisting of two microphones and a coupler volume, is sketched in Figure 1. The coupler has a length L and a radius a. Both are parameters that can be changed in the COMSOL model to fit the dimensions of various standardized couplers. A 2D axisymmetric representation is used on the COMSOL model.

The ratio between the input current I_{in} and the output voltage V_{out} is the electric transfer impedance Z_e (this quantity is measured). The acoustic transfer impedance Z_a of the coupler is the ratio between the average pressure at the receiver boundary $p_{av,r}$ and the volume velocity at the source boundary Q_s . The product of the microphone sensitivity Mof microphone A and microphone B is given as

$$M_{\rm A}M_{\rm B} = \frac{Z_{\rm c}}{Z_{a}}$$
 $Z_{e} = \frac{V_{\rm out}}{I_{\rm in}}$ $Z_{\rm a} = \frac{p_{\rm av,r}}{Q_{\rm s}}$

Using three microphones (A, B, and C), in all combinations of source and receiver, results in a system of three equations with the three sensitivities as unknowns. This system is solved to produce the calibrated microphone sensitivities M_A , M_B , and M_C . In this procedure, it is assumed that the acoustic transfer impedance Z_a of the coupler volume is known. To achieve high precision calibration the acoustic transfer impedance has to be well defined for a large frequency. Material properties of air also need to be well described.

The calibration procedure, acoustic transfer impedance, and material properties are described by the international standard, IEC 61094-2:2009, Ref. 3. As an alternative to the IEC standard, you can choose to model the acoustic transfer impedance in full detail using the **Thermoviscous Acoustics, Frequency Domain** physics interface. Detailed moist air material properties can be computed using the **Thermodynamics** feature. This will yield an acoustic transfer impedance that is correct at all frequencies. The simulation only assumes linear acoustics, while modeling all thermoviscous effects in details. That is, the dynamics of the thermal and viscous boundary layers, as well as the complex transition from adiabatic to isothermal behavior for decreasing frequencies. In the limit of large couplers and high frequencies correct wave propagation behavior is also captured.

The acoustic transfer impedance is computed in this model and compared to three different analytical models, as described below.

ISOTHERMAL LIMIT, VERY LOW FREQUENCY

At very low frequencies the system behaves isothermally (the thermal boundary layers fill the entire coupler volume). The pressure change dp, in the volume V, for a given volume change dV, follows from the definition of isothermal compressibility

$$\beta_{\rm T} = \frac{1}{\rho} \frac{\mathrm{d}\rho}{\mathrm{d}p} \Big|_{T} \qquad \mathrm{d}\rho = -\rho \frac{\mathrm{d}V}{V}$$
$$\mathrm{d}p = -\frac{\mathrm{d}V}{\beta_{\rm T}V}$$

The variables for this expression are located under **Definitions>Variables - Isothermal Limit** (very low frequency).

TRANSMISSION LINE, HIGH FREQUENCY

In the high frequency limit, disregarding boundary layer effects on the source and receiver boundaries, the coupler can be approximated with a transmission line defined by a transfer matrix **T**. The model takes the boundary layer losses at the coupler walls into account through a complex-valued characteristic specific impedance and propagation constant. Details and an expression for the transfer matrix is described in the tutorial Wax Guard Acoustics: Transfer Matrix Computation. The acoustic transfer impedance (when the receiver is sound hard) is the simply defined as $Z_a = 1/T_{21}$. and the admittance is directly given by T_{21} .

The variables for this analytical model are located under **Definitions>Transmission Line (high frequency)**.

MODEL BY VINCENT AND OTHERS, LOW FREQUENCY

More advanced analytical models exist for computing the transfer impedance of a cylinder, including all viscous and thermal effects. In a recent work by Vincent and others (see Ref. 5), the work of Gerber (see Ref. 4) is extended to be valid in the full low frequency range (from medium all the way to very low frequencies and the isothermal limit). The pressure in the coupler and the transfer impedance are given by equations 24, 26, 27, and 30 in Ref. 5. Note that the source and receiver admittances are 0 (perfect source and sound hard wall).

One of the analytical expressions include an infinite sum. In COMSOL, the sum is implemented using the sum() operator, with 10 terms in the sum. The analytical model also includes zeros of the Bessel function, the first 10 used are tabulated in the interpolation function lam_n() defined under **Definitions**.

The variables for this analytical model are located under **Definitions>Vincent et al. (low frequency)**.

Results and Discussion

The root mean square (RMS) velocity in the coupler and the temperature fluctuations, for four evaluation frequencies of 0.1 Hz, 1 Hz, 10 Hz, and 100 Hz, is depicted in Figure 2 and Figure 3, respectively. The dependency of the viscous and thermal boundary layers on frequency and the extent into the geometry, can be seen graphically. In the model, the coupler has a length L of 5 mm and a radius a of 4.5 mm.

The real and imaginary part of the pressure in the coupler is depicted in Figure 4 and Figure 5, respectively. The graphs show the expected correlation between the COMSOL model and the analytical models. The isothermal limit represents the asymptotic behavior for the frequency going to 0. The transmission line model shows the best agreement on the high frequency limit, without giving a perfect match. The model by Vincent and others agrees well with the COMSOL model (within 0.01 dB) on the real part below about 150 Hz and for the imaginary part below about 5 Hz.

The real and imaginary part of the acoustic transfer impedance Z_{a} , is plotted as function of frequency in Figure 6. The graph again shows good correlation between the model by Vincent and others for frequencies below 100 Hz. Having consistent and correct values of the acoustic transfer impedance is essential for high fidelity calibration.

In this tutorial, the moist air material generated using the **Predefined System** option for **Moist air** of the **Thermodynamics** feature is used. This functionality requires the Liquid & Gas Properties Module. Once the mixture is set up using the steps of the wizard a moist air material is generated using the **Generate Material** option. Note that the content of the various species (mole fractions) that make up air, can be modified in the **Local properties** table inside the **Materials>Gas: Moist Air I** material. This allows to model any air variant.

As an example, the dependency on the ambient temperature of the density, the dynamic viscosity, the thermal conductivity, and the speed of sound is depicted in Figure 7. The graphs shows the values for four values of the relative humidity. This dependency is automatically included in the COMSOL model results as **Relative humidity** ϕ_w , is a **Model Input** when the moist air material is used.



Figure 2: Acoustic velocity fluctuations (RMS) in the coupler volume at 0.1, 1, 10, and 100 Hz.



Figure 3: Acoustic temperature fluctuations in the coupler at 0.1, 1, 10, and 100 Hz.



Figure 4: Real part of the pressure in the coupler evaluated using the COMSOL model and the three analytical approximations.



Figure 5: Imaginary part of the pressure in the coupler evaluated using the COMSOL model and the three analytical approximations.



Figure 6: Acoustic transfer impedance for the coupler volume evaluated using the COMSOL model as well as the low and high frequency models.



Figure 7: Various material properties of moist air (density, dynamic viscosity, thermal conductivity, and speed of sound) evaluated as function of temperature (from 0 to 50° C) for four different values of the relative humidity (0.0, 0.2, 0.4, and 0.8).

Notes About the COMSOL Implementation

BULK LOSSES

When modeling fluids with the thermoviscous acoustics interface, the damping and dissipation due to the thermal and viscous boundary layers is included in full detail. In the bulk/volume of the fluid (away from the boundaries) the model captures the attenuation that corresponds to the classical thermoviscous attenuation α_{tv} . This is not the correct bulk attenuation as it does not include losses due to relaxation. These losses that are, for example, captured when using the **Atmosphere** attenuation model in pressure acoustics. In most microacoustic applications the boundary layer losses are orders of magnitude higher than the bulk losses, so it is not important. In the present coupler application, the bulk losses may start to play a small role at very high frequencies. In this case, the true bulk

losses may be included by defining a frequency dependent bulk viscosity $\mu_{\rm B} = \mu_{\rm B}(f)$, according to

$$\mu_{\rm B}(f) = \frac{2\rho c^3}{\omega^2} \alpha(T_0, p_0, \phi_{\rm w}) - \mu \left(\frac{4}{3} + \frac{\gamma - 1}{\Pr}\right)$$

where $\omega = 2\pi f$ and α is the true atmosphere attenuation. This will introduce consistent bulk/volume losses that also include relaxation processes and other mechanisms included in the expression for α .

MEMBRANE DEFORMATION

In this model, plane wave propagation is assumed. For certain larger couplers, the effects of the true membrane deformation can start to play a role a high frequencies. In this case, the membrane can be included in the model. See for example The Brüel & Kjær 4134 Condenser Microphone tutorial.

MESH

The mesh used in this model uses two boundary layer meshes. The first resolves the physics by resolving the thermal and viscous boundary layer thickness (also known as the penetration depth). The second adds a small single layer that is used to resolve the numerical singularities at the corners of the geometry.

References

1. Danish Primary Laboratory of Acoustic Microphone Reciprocity Calibration Calculation Program for Reciprocity Calibration, Technical Review No. 1-1998, Brüel & Kjær, Denmark.

2. E. Frederiksen, "Acoustic metrology - an overview of calibration methods and their uncertainty," *Int. J. Metrol. Qual. Eng.*, vol. 4, pp. 97–107, 2013.

3. International Standard, *Electroacoustics - Measurement microphones - Part 2: Primary method for pressure calibration of laboratory standard microphones by the reciprocity technique*, IEC 61094-2:2009.

4. H. Gerber, "Acoustic Properties of Flui.Filled Chambers at Infrasonic Frequencies in the Absence of Convection," *J. Acoust. Soc. Am.*, vol. 36, p. 1427, 1964.

5. P. Vincent, D. Rodrigues, F. Larsonnier, C. Guianvarc'h, and S. Durand, "Acoustic transfer admittance of cylindrical cavities in infrasonic frequency range," *Metrologia*, vol. 56, p. 015003, 2019.

Application Library path: Acoustics_Module/Tutorials,

_Thermoviscous_Acoustics/pressure_reciprocity_calibration_coupler

Modeling Instructions

From the File menu, choose New.

NEW

In the New window, click 🔗 Model Wizard.

MODEL WIZARD

- I In the Model Wizard window, click 🚈 2D Axisymmetric.
- 2 In the Select Physics tree, select Acoustics>Thermoviscous Acoustics> Thermoviscous Acoustics, Frequency Domain (ta).
- 3 Click Add.
- 4 Click \ominus Study.
- 5 In the Select Study tree, select General Studies>Frequency Domain.
- 6 Click 🗹 Done.

GLOBAL DEFINITIONS

Parameters - Physics

- I In the Model Builder window, under Global Definitions click Parameters I.
- 2 In the Settings window for Parameters, type Parameters Physics in the Label text field.
- 3 Locate the Parameters section. Click 📂 Load from File.
- 4 Browse to the model's Application Libraries folder and double-click the file pressure_reciprocity_calibration_coupler_parameters_physics.txt.

Parameters - Geometry

- I In the Home toolbar, click P; Parameters and choose Add>Parameters.
- 2 In the Settings window for Parameters, type Parameters Geometry in the Label text field.
- **3** Locate the **Parameters** section. Click *b* Load from File.

4 Browse to the model's Application Libraries folder and double-click the file pressure_reciprocity_calibration_coupler_parameters_geometry.txt.

GEOMETRY I

Rectangle 1 (r1)

- I In the Geometry toolbar, click Rectangle.
- 2 In the Settings window for Rectangle, locate the Size and Shape section.
- **3** In the **Width** text field, type a.
- **4** In the **Height** text field, type L.
- 5 Click 🟢 Build All Objects.

ADD MATERIAL

- I In the Home toolbar, click 🙀 Add Material to open the Add Material window.
- 2 Go to the Add Material window.
- 3 In the tree, select Built-in>Air.
- 4 Click Add to Component in the window toolbar.
- 5 In the Home toolbar, click 🙀 Add Material to close the Add Material window.

MATERIALS

Air (dry)

I In the Settings window for Material, type Air (dry) in the Label text field.

You just added the default air material from the **Material Library**. This is a model of dry air where certain properties like density and speed of sound are based on the ideal gas law. A detailed moist air material can be conveniently set up using a **Predefined System** defined by the **Thermodynamics** feature.

Setting up the moist air material requires the Liquid & Gas Properties Module. If you do not have that product, you can skip the next steps and use the **Air (dry)** material instead.

GLOBAL DEFINITIONS

In the Physics toolbar, click 🖄 Thermodynamics and choose Predefined System.

SELECT SYSTEM

- I Go to the Select System window.
- 2 From the Predefined thermodynamic system list, choose Moist air.

3 Click Next in the window toolbar.

SELECT SPECIES

- I Go to the Select Species window.
- 2 Click Next in the window toolbar.

SELECT THERMODYNAMIC MODEL

- I Go to the Select Thermodynamic Model window.
- 2 From the Gas phase model list, choose Peng-Robinson.

The choice of **Gas phase model** depends on the exterior conditions like ambient pressure and temperature. The different models are described in the *Liquid & Gas Properties Module User's Guide* in the Thermodynamics Models section.

3 Click **Finish** in the window toolbar.

GLOBAL DEFINITIONS

Moist Air I (ppI)

In the Model Builder window, under Global Definitions>Thermodynamics right-click Moist Air I (ppI) and choose Generate Material.

SELECT SPECIES

- I Go to the Select Species window.
- 2 Click Next in the window toolbar.

SELECT PROPERTIES

- I Go to the Select Properties window.
- 2 Click Next in the window toolbar.

DEFINE MATERIAL

- I Go to the Define Material window.
- 2 Click **Finish** in the window toolbar.

Note that the bulk viscosity is defined in terms of the dynamic viscosity. The relation is the same as in the default (dry) air material. Values of the bulk viscosity are experimentally obtained using high-frequency absorption techniques. More details can be found in the Acoustic Properties of Fluids chapter of the *Acoustics Module User's Guide*.

Use the moist air material for the model.

MATERIALS

Gas: Moist Air I (ppImatl)

- I In the Model Builder window, under Component I (compl)>Materials click Gas: Moist Air I (pplmatl).
- **2** Select Domain 1 only.

DEFINITIONS

Variables - Material Properties

- I In the Model Builder window, expand the Component I (compl)>Definitions node.
- 2 Right-click **Definitions** and choose **Variables**.
- **3** In the **Settings** window for **Variables**, type **Variables Material Properties** in the **Label** text field.
- 4 Locate the Variables section. Click 📂 Load from File.
- 5 Browse to the model's Application Libraries folder and double-click the file pressure_reciprocity_calibration_coupler_variables_material.txt.

Variables - Isothermal Limit (very low frequency)

- I Right-click **Definitions** and choose **Variables**.
- 2 In the Settings window for Variables, type Variables Isothermal Limit (very low frequency) in the Label text field.
- **3** Locate the Variables section. Click *b* Load from File.
- **4** Browse to the model's Application Libraries folder and double-click the file pressure_reciprocity_calibration_coupler_variables_isothermal.txt.

Variables - Transmission Line (high frequency)

- I Right-click **Definitions** and choose **Variables**.
- 2 In the Settings window for Variables, type Variables Transmission Line (high frequency) in the Label text field.
- 3 Locate the Variables section. Click 📂 Load from File.
- 4 Browse to the model's Application Libraries folder and double-click the file pressure_reciprocity_calibration_coupler_variables_transmission.txt.

Variables - Vincent et al. (low frequency)

- I Right-click **Definitions** and choose **Variables**.
- 2 In the Settings window for Variables, type Variables Vincent et al. (low frequency) in the Label text field.

- **3** Locate the Variables section. Click *b* Load from File.
- 4 Browse to the model's Application Libraries folder and double-click the file pressure_reciprocity_calibration_coupler_variables_vincent.txt.

Integration 1 (intop1)

- I In the Definitions toolbar, click 🖉 Nonlocal Couplings and choose Integration.
- 2 In the Settings window for Integration, type intop_s in the Operator name text field.
- **3** Locate the **Source Selection** section. From the **Geometric entity level** list, choose **Boundary**.
- 4 Select Boundary 2 only.

Integration 2 (intop2)

- I In the Definitions toolbar, click 🖉 Nonlocal Couplings and choose Integration.
- 2 In the Settings window for Integration, type intop_r in the Operator name text field.
- **3** Locate the **Source Selection** section. From the **Geometric entity level** list, choose **Boundary**.
- 4 Select Boundary 3 only.

Integration 3 (intop3)

- I In the Definitions toolbar, click 🖉 Nonlocal Couplings and choose Integration.
- 2 In the Settings window for Integration, type intop_pnt in the Operator name text field.
- 3 Locate the Source Selection section. From the Geometric entity level list, choose Point.
- **4** Select Point 4 only.
- 5 Locate the Advanced section. Clear the Compute integral in revolved geometry check box.

Interpolation 1 (int1)

- I In the **Definitions** toolbar, click **Interpolation**.
- 2 In the Settings window for Interpolation, locate the Definition section.
- 3 From the Data source list, choose File.
- 4 Click 📂 Browse.
- 5 Browse to the model's Application Libraries folder and double-click the file pressure_reciprocity_calibration_coupler_bessel_zeros.txt.
- 6 Click **[III]** Import.
- 7 In the Function name text field, type lam_n.
- 8 Locate the Interpolation and Extrapolation section. From the Interpolation list, choose Nearest neighbor.

9 Locate the Units section. In the Argument table, enter the following settings:

Argument	Unit
t	1

IO In the **Function** table, enter the following settings:

Function	Unit
lam_n	1

II Click the 💿 Show More Options button in the Model Builder toolbar.

12 In the Show More Options dialog box, in the tree, select the check box for the node Physics>Advanced Physics Options.

I3 Click OK.

THERMOVISCOUS ACOUSTICS, FREQUENCY DOMAIN (TA)

Thermoviscous Acoustics Model I

- I In the Model Builder window, under Component I (compl)>Thermoviscous Acoustics, Frequency Domain (ta) click Thermoviscous Acoustics Model I.
- **2** In the **Settings** window for **Thermoviscous Acoustics Model**, locate the **Model Input** section.
- **3** In the ϕ_w text field, type relH.

Because the relative humidity is set up as a **Model Input** for the moist air material, it automatically appears as an input in the physics.

Velocity I

- I In the Physics toolbar, click Boundaries and choose Velocity.
- **2** Select Boundary 2 only.
- 3 In the Settings window for Velocity, locate the Velocity section.
- 4 Select the Prescribed in r direction check box.
- **5** Select the **Prescribed in z direction** check box.
- **6** In the u_{0z} text field, type ta.iomega*dn.
- 7 Click to expand the **Excluded Points** section. Select Point 3 only.

Isothermal I

- I In the Physics toolbar, click Boundaries and choose Isothermal.
- **2** Select Boundary 2 only.

In this model, the mesh is set up manually. Proceed by directly adding the desired mesh component.

MESH I

Free Triangular 1

In the **Mesh** toolbar, click **Free Triangular**.

Size

- I In the Model Builder window, click Size.
- 2 In the Settings window for Size, locate the Element Size section.
- 3 Click the **Custom** button.
- **4** Locate the **Element Size Parameters** section. In the **Maximum element size** text field, type a/10.
- 5 In the Minimum element size text field, type dvisc0.

Size I

- I In the Model Builder window, right-click Free Triangular I and choose Size.
- 2 In the Settings window for Size, locate the Geometric Entity Selection section.
- **3** From the Geometric entity level list, choose Boundary.
- **4** Select Boundaries 2–4 only.
- 5 Locate the **Element Size** section. Click the **Custom** button.
- 6 Locate the Element Size Parameters section.
- 7 Select the Maximum element size check box. In the associated text field, type dvisc0.

Boundary Layers 1

- I In the Mesh toolbar, click Boundary Layers.
- 2 In the Settings window for Boundary Layers, click to expand the Transition section.
- **3** Clear the **Smooth transition to interior mesh** check box.

Boundary Layer Properties

- I In the Model Builder window, click Boundary Layer Properties.
- 2 Select Boundary 4 only.
- 3 In the Settings window for Boundary Layer Properties, locate the Layers section.
- 4 From the Thickness specification list, choose First layer.
- 5 In the **Thickness** text field, type 0.2*dvisc.

Boundary Layers 2

- I In the Mesh toolbar, click Moundary Layers.
- 2 In the Settings window for Boundary Layers, locate the Transition section.
- 3 Clear the Smooth transition to interior mesh check box.

Boundary Layer Properties

- I In the Model Builder window, click Boundary Layer Properties.
- **2** Select Boundaries 2 and 3 only.
- 3 In the Settings window for Boundary Layer Properties, locate the Layers section.
- **4** From the **Thickness specification** list, choose **First layer**.
- 5 In the **Thickness** text field, type 0.2*dvisc.

Boundary Layers 3

- I In the Mesh toolbar, click Boundary Layers.
- 2 In the Settings window for Boundary Layers, locate the Transition section.
- **3** Clear the **Smooth transition to interior mesh** check box.

Boundary Layer Properties

- I In the Model Builder window, click Boundary Layer Properties.
- **2** Select Boundary 4 only.
- 3 In the Settings window for Boundary Layer Properties, locate the Layers section.
- 4 In the Number of layers text field, type 1.
- 5 From the Thickness specification list, choose First layer.
- 6 In the Thickness text field, type 2e-6.

Boundary Layers 4

- I In the Mesh toolbar, click M Boundary Layers.
- 2 In the Settings window for Boundary Layers, locate the Transition section.
- **3** Clear the **Smooth transition to interior mesh** check box.

Boundary Layer Properties

- I In the Model Builder window, click Boundary Layer Properties.
- 2 Select Boundaries 2 and 3 only.
- 3 In the Settings window for Boundary Layer Properties, locate the Layers section.
- 4 In the Number of layers text field, type 1.
- 5 From the Thickness specification list, choose First layer.

- 6 In the Thickness text field, type 2e-6.
- 7 Click 📗 Build All.

STUDY I

Step 1: Frequency Domain

- I In the Model Builder window, under Study I click Step I: Frequency Domain.
- 2 In the Settings window for Frequency Domain, locate the Study Settings section.
- 3 Click Range.
- 4 In the Range dialog box, choose ISO preferred frequencies from the Entry method list.
- 5 In the Start frequency text field, type 0.05.
- 6 In the **Stop frequency** text field, type 10000.
- 7 From the Interval list, choose 1/3 octave.
- 8 Click Replace.
- **9** In the **Home** toolbar, click **= Compute**.

RESULTS

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Acoustic Pressure (ta)
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RMS Acoustic Velocity (ta)

- I In the Model Builder window, under Results click Acoustic Velocity (ta).
- 2 In the Settings window for 2D Plot Group, type RMS Acoustic Velocity (ta) in the Label text field.

Surface

- I In the Model Builder window, expand the RMS Acoustic Velocity (ta) node, then click Surface.
- 2 In the Settings window for Surface, locate the Expression section.
- 3 In the Expression text field, type ta.v_rms.



4 In the RMS Acoustic Velocity (ta) toolbar, click 💿 Plot.

Temperature Variation (ta)



Pressure in Coupler: real(p)

- I In the Home toolbar, click 🚛 Add Plot Group and choose ID Plot Group.
- 2 In the Settings window for ID Plot Group, type Pressure in Coupler: real(p) in the Label text field.
- 3 Click to expand the Title section. From the Title type list, choose Label.
- 4 Locate the Plot Settings section.
- **5** Select the **x-axis label** check box. In the associated text field, type **f** (Hz).
- 6 Select the y-axis label check box. In the associated text field, type Pressure (Pa).
- 7 Locate the Legend section. From the Position list, choose Upper left.

Point Graph 1

- I Right-click Pressure in Coupler: real(p) and choose Point Graph.
- **2** Select Point 2 only.
- 3 In the Settings window for Point Graph, locate the y-Axis Data section.
- **4** In the **Expression** text field, type real(ta.p_t).
- 5 Click to expand the Legends section. Select the Show legends check box.
- 6 From the Legends list, choose Manual.

7 In the table, enter the following settings:

Legends

COMSOL model

Global I

- I In the Model Builder window, right-click Pressure in Coupler: real(p) and choose Global.
- 2 In the Settings window for Global, locate the y-Axis Data section.
- **3** In the table, enter the following settings:

Expression	Unit	Description
real(dpT)	Ра	Isothermal limit (very low frequency)
real(i*omega*dV/Y_tl)	Ра	Transmission line (high frequency)
real(p_galf)	Ра	Vincent et al. (low frequency)

4 In the **Pressure in Coupler: real(p)** toolbar, click **O** Plot.

- **5** Click the **x-Axis Log Scale** button in the **Graphics** toolbar.
- 6 Click the **y-Axis Log Scale** button in the **Graphics** toolbar.



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Pressure in Coupler: imag(p)

- I Right-click Pressure in Coupler: real(p) and choose Duplicate.
- 2 In the Settings window for ID Plot Group, type Pressure in Coupler: imag(p) in the Label text field.
- 3 Locate the Legend section. From the Position list, choose Upper right.

Point Graph 1

- I In the Model Builder window, expand the Pressure in Coupler: imag(p) node, then click Point Graph I.
- 2 In the Settings window for Point Graph, locate the y-Axis Data section.
- **3** In the **Expression** text field, type imag(ta.p_t).

Global I

- I In the Model Builder window, click Global I.
- 2 In the Settings window for Global, locate the y-Axis Data section.
- **3** In the table, enter the following settings:

Expression	Unit	Description
<pre>imag(dpT)</pre>	Ра	Isothermal limit (very low frequency)
<pre>imag(i*omega*dV/Y_tl)</pre>	Ра	Transmission line (high frequency)
<pre>imag(p_galf)</pre>	Ра	Vincent et al. (low frequency)

4 In the Pressure in Coupler: imag(p) toolbar, click **I** Plot.



5 Click the **y-Axis Log Scale** button in the **Graphics** toolbar.

Acoustic Transfer Impedance: real(Z) and imag(Z)

- I In the Home toolbar, click 🚛 Add Plot Group and choose ID Plot Group.
- 2 In the Settings window for ID Plot Group, type Acoustic Transfer Impedance: real(Z) and imag(Z) in the Label text field.
- 3 Locate the Title section. From the Title type list, choose Label.
- 4 Locate the Plot Settings section.
- 5 Select the x-axis label check box. In the associated text field, type f (Hz).
- 6 Select the y-axis label check box. In the associated text field, type Z_{a, 12} (kg/(m⁴s)).
- 7 Locate the Axis section. Select the x-axis log scale check box.
- 8 Select the y-axis log scale check box.
- 9 Locate the Legend section. From the Position list, choose Lower left.

Global I

- I Right-click Acoustic Transfer Impedance: real(Z) and imag(Z) and choose Global.
- 2 In the Settings window for Global, locate the y-Axis Data section.

3 In the table, enter the following settings:

Expression	Unit	Description
<pre>real((intop_s(ta.p_t)/S)/ (ta.iomega*dn*S))</pre>	kg/(m^4*s)	real(Z), COMSOL model
-imag((intop_s(ta.p_t)/ S)/(ta.iomega*dn*S))	kg/(m^4*s)	-imag(Z), COMSOL model

4 Click to expand the Coloring and Style section. From the Width list, choose 2.

Global 2

- I In the Model Builder window, right-click Acoustic Transfer Impedance: real(Z) and imag(Z) and choose Global.
- 2 In the Settings window for Global, locate the y-Axis Data section.
- **3** In the table, enter the following settings:

Expression	Unit	Description
real(1/Y_tl)	kg/(m^4*s)	real(Z), Transmission line
-imag(1/Y_tl)	kg/(m^4*s)	-imag(Z), Transmission line

- **4** Locate the **Coloring and Style** section. Find the **Line style** subsection. From the **Line** list, choose **Dashed**.
- 5 From the Color list, choose Cycle (reset).

Global 3

- I Right-click Acoustic Transfer Impedance: real(Z) and imag(Z) and choose Global.
- 2 In the Settings window for Global, locate the y-Axis Data section.
- **3** In the table, enter the following settings:

Expression	Unit	Description	
real(1/Ya)	kg/(m^4*s)	real(Z), Vincent et al.	
-imag(1/Ya)	kg/(m^4*s)	-imag(Z), Vincent et al.	

- **4** Locate the **Coloring and Style** section. Find the **Line style** subsection. From the **Line** list, choose **None**.
- **5** From the **Color** list, choose **Cycle (reset)**.
- 6 Find the Line markers subsection. From the Marker list, choose Point.
- 7 From the **Positioning** list, choose **Interpolated**.
- 8 In the Number text field, type 25.



9 In the Acoustic Transfer Impedance: real(Z) and imag(Z) toolbar, click on Plot.

Next create a grid dataset that will be used to plot the material properties as function of temperature for several values of the relative humidity.

Grid ID I

- I In the **Results** toolbar, click **More Datasets** and choose **Grid>Grid ID**.
- 2 In the Settings window for Grid ID, locate the Parameter Bounds section.
- **3** In the **Name** text field, type Tg.
- 4 In the Minimum text field, type 273.15.
- 5 In the Maximum text field, type 323.15.

Density

- I In the Results toolbar, click \sim ID Plot Group.
- 2 In the Settings window for ID Plot Group, type Density in the Label text field.
- 3 Locate the Data section. From the Dataset list, choose Grid ID I.
- 4 From the Parameter selection (freq) list, choose First.
- 5 Locate the Title section. From the Title type list, choose Label.
- 6 Locate the Plot Settings section.

- 7 Select the x-axis label check box. In the associated text field, type Temperature (^oC).
- 8 Select the y-axis label check box. In the associated text field, type Density (kg/m³).

Line Graph I

- I Right-click **Density** and choose **Line Graph**.
- 2 In the Settings window for Line Graph, locate the y-Axis Data section.
- 3 In the Expression text field, type subst(pp1mat1.def.rho,minput.T,Tg[K/m], minput.pA,p0,minput.phi,0).
- 4 Click to expand the Legends section. Select the Show legends check box.
- 5 From the Legends list, choose Manual.
- 6 In the table, enter the following settings:

Legends

relH = 0.0

7 In the **Density** toolbar, click **I** Plot.

Line Graph 2

- I Right-click Line Graph I and choose Duplicate.
- 2 In the Settings window for Line Graph, locate the y-Axis Data section.
- 3 In the Expression text field, type subst(pp1mat1.def.rho,minput.T,Tg[K/m], minput.pA,p0,minput.phi,0.2).
- 4 Locate the Legends section. In the table, enter the following settings:

Legends

relH = 0.2

5 In the **Density** toolbar, click **O** Plot.

Line Graph 3

- I Right-click Line Graph 2 and choose Duplicate.
- 2 In the Settings window for Line Graph, locate the y-Axis Data section.
- 3 In the Expression text field, type subst(pp1mat1.def.rho,minput.T,Tg[K/m], minput.pA,p0,minput.phi,0.4).

4 Locate the Legends section. In the table, enter the following settings:

Legends

relH = 0.4

5 In the **Density** toolbar, click **I** Plot.

Line Graph 4

- I Right-click Line Graph 3 and choose Duplicate.
- 2 In the Settings window for Line Graph, locate the y-Axis Data section.
- 3 In the Expression text field, type subst(pp1mat1.def.rho,minput.T,Tg[K/m], minput.pA,p0,minput.phi,0.8).
- 4 Locate the Legends section. In the table, enter the following settings:

Legends

relH = 0.8

5 In the **Density** toolbar, click **I** Plot.



Duplicate the density plot in order to create plots of the dynamic viscosity (pp1mat1.def.mu), thermal conductivity (pp1mat1.def.k_iso), and speed of sound

(pp1mat1.def.c). Finally, group the plots in a **Node Group**. All the plots are depicted in the Results and Discussion section of the documentation.