

# Lumped Receiver with Full Vibroacoustic Coupling

# Introduction

When simulations are involved in the development of mobile devices, consumer electronics, hearing aids, or headsets, it is necessary to consider how the transducers interact with the rest of the system. Here, we will show the analysis of the interaction between a vibration isolation mounting and a miniature hearing aid transducer using a lumped representation of the transducer. The lumped model is simplified as an equivalent electroacoustic circuit. The vibration and acoustic characteristics of the lumped model are then coupled to a multiphysics model of the vibration isolation system to achieve a full system analysis.

In this model, the miniature hearing aid transducer is a Knowles<sup>TM</sup> TEC-30033 balanced armature receiver, a miniature loudspeaker commonly used in high performance hearing aid packages. A common method of vibration isolation of a receiver is to attach it to the free end of a cantilevered tube. The tube channels the sound to the ear tips and the ear canal and at the same time it reduces the vibration energy transmitted back to the hearing aid package, see Figure 1. This model replicates a test setup that consists of a silicone tube of length 9 mm and inner diameter 1 mm that is attached to a 2 cc coupler, a common cavity of 2 cm<sup>3</sup> utilized as an acoustic load. The setup is illustrated in Figure 5.

The receiver's internal electrical, magnetic, mechanical, and acoustic properties are linearized and represented with an electrical network topology<sup>1</sup>. The mechanical forces within the network are probed and applied as rigid body loads to the receiver. The output acoustic pressure and probed rigid body motion of the receiver are coupled to a finite element (FEM) model of the silicone tubing attachment and acoustic coupler. The thermoviscous losses in the narrow tubing are included using the homogenized approach offered by the *Narrow Region Acoustics* feature in the *Pressure Acoustics, Frequency Domain* interface.

The simulated acoustic response measured in the coupler and the vibration characteristics obtained in the model are compared to measurements. The acoustic response is obtained from the coupler microphone and the vibration characteristics from laser vibrometer measurements.

The model shows how to set up the coupling between the equivalent circuit model (a SPICE representation) and the rigid domain used to model the receiver. This approach allows a full system simulation of, for example, a hearing aid without the need of a detailed receiver model. This can be used to study the full vibroacoustic feedback path between the receiver (miniature loudspeaker) and microphones.

<sup>1.</sup> This model was created based upon data provided by Knowles Electronics LLC, Illinois USA.

**Note:** This model is an extension of the Lumped Receiver Connected to Test Setup with a 0.4-cc Coupler tutorial where only the acoustics are considered. The current tutorial also considers a different model of balanced armature receiver.

**Note:** This model requires the Acoustics Module, the Structural Mechanics Module, and the AC/DC Module.



Figure 1: (left) Schematic representation of the vibration feedback path from the receiver (miniature loudspeaker) to the microphones in a behind-the-ear (BTE) hearing aid. (right) Schematic of how a BTE is placed on the human ear including the vent where sound leaks.

# Model Definition

Miniature loudspeaker and other transducers are used in many modern consumer electronics products like smart phones, ear buds, tablets, and hearing aids. In most of these applications, it is desirable to optimize the sound quality and miniaturize the product. For certain applications, like hearing aids, the maximal output level is also important. In all cases, understanding the acoustic and vibrational behavior is important in order to prevent, for example, feedback effects. The specific example of the integration of a balanced armature receiver (or simply receiver, the name given for the miniature loudspeaker in hearing aids) into a behind-the-ear or BTE hearing aid is schematically depicted in Figure 1. The figure shows a cross section of the hearing aid, location of the transducers, earmold tubing, and the possible feedback path. The feedback path is generated by either the sound in the tubing (generating vibrations that couple to the microphones), directly as mechanical vibrations (red arrows), or by all acoustic when sound is transmitted through the earmold tube, ear canal, and vent to the microphones. These different feedback path need to be understood an possibly isolated.

A fully detailed multiphysics vibro-electro-acoustic model of a miniature transducer is in itself very complex. A rendering of a balanced armature receiver type transducer can be seen in Figure 2. This means that the task of understanding its system integration can easily become computationally expensive if all physics are modeled in detail with a FEM model. That is why a lumped representation of both the electroacoustic behavior and the vibration characteristics of the transducers is desirable to enable a full system simulation.

Specifically in this tutorial, a Knowles<sup>TM</sup> TEC-30033 receiver is modeled by a lumped equivalent circuit (see Figure 4) coupled to the motion of a rigid body domain with the mass properties of the transducer (see Figure 3). The rigid domain is characterized by its center of mass  $\mathbf{X}_{cm}$  and moment of inertia **I**. Both can be extracted from a detailed CAD drawing of the transducer. In COMSOL, this can be done using the **Mass Properties** feature found under the **Definitions** node.



Figure 2: Rendering of the inner structure of a balanced armature receiver, © Knowles Electronics LLC. Detailed modeling of the system is very computationally demanding and requires coupling electromagnetic fields, structural vibrations and acoustics including thermoviscous losses. Image courtesy of Knowles Electronics LLC, Illinois USA.

Since the orientation and location of the receiver can be arbitrary in a full system simulation, its mechanical properties are given with respect to the geometric center  $\mathbf{X}_g$  of the receiver box and local orientation of the receiver. The geometric center of the rigid domain corresponding to the receiver is calculated in the model using the **Mass Properties** feature with a unit density expression. The orientation of the transducer is given by setting up a **Base Vector System** defined through a geometry **Work Plane** (placed on top of the receiver box). The work plane and local coordinate system can be seen in Figure 3. Using this approach, the location and orientation of the transducer is easily defined.

The location of the center of mass  $\mathbf{X}_{cm}$  in the global coordinate system is then given by

$$\mathbf{X}_{\rm cm} = \mathbf{X}_{\rm g} + \left[T_{ij}\right]^{-1} \mathbf{X}_{\rm cm}^{\rm local} \tag{1}$$

where  $\mathbf{X}_{cm}^{local}$  is the center-of-mass in the local receiver system of coordinates and  $[T_{ij}]^{-1}$  is the coordinate transformation matrix. The transformation matrix is automatically defined by the **Base Vector System** feature. The components are given by the variables sys2.invT11, sys2.invT12, sys2.invT13, and so on. The center-of-mass coordinates are defined as variables under the **Receiver Variables** node in the **Definitions**.



Figure 3: Local coordinate system defined by adding a work plane at the receiver box surface. In the inset, a schematic of the applied forces and moment on the receiver, © Knowles Electronics LLC. These are due to the movement of the armature and diaphragm (see Figure 2). Image courtesy of Knowles Electronics LLC, Illinois USA.

The circuit model topology of the transducer is depicted in Figure 4. The equivalent circuit network is imported as a **Subcircuit Definition** in the **Electrical Circuit** interface. Such a network is capable to capture the electro-acoustic performance of most balanced armature receivers produced by Knowles. The network represents the electro-magnetic, mechanical, and acoustic parts of the receiver (different colors in the diagram). The acoustics in the circuit are bidirectionally coupled to the finite element domain using the **Circuit** connection option of the **Lumped Port** condition.



Figure 4: Lumped circuit representation of the balanced armature receiver, © Knowles Electronics LLC. Note that the Karm component is a semi capacitor in this schematic. In the COMSOL implementation it is replaced by a resistor with a frequency dependent resistance. Image courtesy of Knowles Electronics LLC, Illinois USA.

The vibration characteristics of the receiver are modeled by applying a force and moment to the center of mass of the rigid domain. The mechanical vibration coupling is only active one way, since the influence of external vibrations is low under normal operating conditions. On the other hand, the acoustics has to be bidirectionally coupled. The applied forces and moment (in the local receiver coordinate system) can be seen in the inset of Figure 3. The values are given by the variables Fx, Fz, and My, also defined in the **Receiver Variables**. The values are defined by:

They relate the voltages in the mechanical part of the spice system to the external forces and moment. The proportionality constants Fx1, Fx2, Fz1, Fz2, My1, and My2 are defined under the **Parameters** and are unique to each receiver model.

# Results and Discussion

The simulated system corresponds to the actual vibration isolation test setup depicted in Figure 5. The system consists of the TEC-30033 receiver, the flexible tubing, and the 2 cc coupler volume. In the experiment, the pressure response is measured by the measurement microphone in the coupler and the vibrations of the transducer are measured using a laser

vibrometer. The simulated pressure and vibration response are compared with experimental data.



Figure 5: Experimental setup consisting of the TEC receiver, earmold tubing, and 2 cc coupler, © Knowles Electronics LLC. The acoustic response is measured by the microphone located in the coupler and the vibrations of the receiver are measured by a laser vibrometer. Image courtesy of Knowles Electronics LLC, Illinois USA.

The sound pressure level response in the coupler is depicted in Figure 6. The agreement between the measurements and the COMSOL simulation is good. Note that the measurements only have been done from 100 Hz up to 10 kHz. Discrepancies at the highest frequencies are expected since the lumped transducer representation is not fully valid at the highest frequencies.



Figure 6: Sound pressure level at the microphone. Comparison of the simulation results (blue curve) and the measurements results (green curve).

The vibration response, defined by the velocity amplitude in the local x and z directions, is depicted in Figure 7 and Figure 8. The measured data includes two measurement series that were performed independently by two groups in the hearing aid industry. The results show good agreement, but also indicate the sensitivity in the measurements. Small changes in the actual earmold tube length or variations the material properties of the silicone tubing, can change both the amplitude and the location of resonances. This type of sensitivity can be studied using simulations by changing the geometry or material parameters.

The velocity response in the local y direction is depicted in Figure 9. Because of the orientation of the model and the applied forces the values are considered to be numerical noise. Notice the low values in the dB scale.



Figure 7: Vibration velocity in the (local) x direction (see inset in Figure 3). Comparison between the simulation results and two independent measurements series.



Figure 8: Vibration velocity in the (local) z direction (see inset in Figure 3). Comparison between the simulation results and two independent measurements series.



Figure 9: Vibration velocity in the (local) y direction. The values are so low that this basically corresponds to numerical noise.

The displacement of the transducer and the earmold tubing, the pressure distribution in the earmold tubing and the coupler, as well as the sound pressure level distribution, is depicted at the frequencies 10 Hz, 100 Hz, 1 kHz, and 10 kHz in Figure 10, Figure 11, and Figure 12, respectively. Detailed analysis of the frequency characteristics at the other studied frequencies can be seen in the model where the system is solved from 10 Hz to 10 kHz in 1/12 th octave steps.



Figure 10: Displacement of the receiver and earmold tubing for 10 Hz, 100 Hz, 1 kHz, and 10 kHz.



Figure 11: Pressure distribution in the earmold tubing and coupler volume for 10 Hz, 100 Hz, 1 kHz, and 10 kHz.



Figure 12: Sound pressure level distribution in the earmold tubing and coupler volume for 10 Hz, 100 Hz, 1 kHz, and 10 kHz.

# Notes About the COMSOL Implementation

- In this model, the viscous and thermal losses associated with the acoustic boundary layer in the narrow tube are modeled using the Narrow Region Acoustics feature in Pressure acoustics. The computational cost is low compared to a full thermoviscous acoustics model and for long structures of constant cross section the losses are exact. However, for complex geometrical structures, the Thermoviscous Acoustics interfaces should be used. Note also that the losses associated with the impedance jump from the narrow tube to the coupler are not included.
- The Electrical Circuit interface uses electrical units. Conversion from electrical to acoustic lumped units are performed automatically in the Lumped port feature with the necessary units. For example, a voltages representing the acoustic pressure at the transducer inlet is transformed to volts, resulting in correct equivalent electric units volts.

In the lumped equivalent circuit model of the receiver, the effects of variation in the skin depth of eddy currents in the steel armature is approximated by a semi-capacitor, a special component with a complex admittance proportional to the square root of *i*ω. In the imported SPICE circuit equivalent topology network list (in the model the file lumped\_receiver\_vibroacoustic\_TEC30033.cir is imported), the value of this component, here a resistor, is temporarily set to 1, using:

RKarm KN020 KN040 1

Then the correct value for this component is entered manually, as a formula, to fit the COMSOL notation:

1[ohm]/(G\_arm\*sqrt(i\*2\*pi\*freq[1/Hz]))

Where G\_arm is a constant valued gain parameter.

**Application Library path:** Acoustics\_Module/Electroacoustic\_Transducers/ lumped\_receiver\_vibroacoustic

# Modeling Instructions

These are the modeling instructions for setting up the model, solving, and creating the postprocessing plots. The Geometry Modeling Instructions are located in the last section at the end of this document.

From the File menu, choose New.

## NEW

In the New window, click Solution Model Wizard.

## MODEL WIZARD

- I In the Model Wizard window, click 间 3D.
- 2 In the Select Physics tree, select AC/DC>Electrical Circuit (cir).
- 3 Click Add.
- 4 In the Select Physics tree, select Structural Mechanics>Solid Mechanics (solid).
- 5 Click Add.
- 6 In the Select Physics tree, select Acoustics>Pressure Acoustics>Pressure Acoustics, Frequency Domain (acpr).
- 7 Click Add.

8 Click 🔿 Study.

 ${\bf 9}~$  In the Select Study tree, select General Studies>Frequency Domain.

10 Click 🗹 Done.

## GLOBAL DEFINITIONS

#### Parameters 1

- I In the Model Builder window, under Global Definitions click Parameters I.
- 2 In the Settings window for Parameters, locate the Parameters section.
- **3** Click **b** Load from File.
- **4** Browse to the model's Application Libraries folder and double-click the file lumped\_receiver\_vibroacoustic\_parameters.txt.

#### GEOMETRY I

- I In the Geometry toolbar, click Insert Sequence and choose Insert Sequence.
- 2 Browse to the model's Application Libraries folder and double-click the file lumped\_receiver\_vibroacoustic\_geom\_sequence.mph.
- 3 In the Geometry toolbar, click 🟢 Build All.

## Work Plane 3 (wp3)

- I In the Geometry toolbar, click 🖶 Work Plane.
- 2 In the Settings window for Work Plane, locate the Plane Definition section.
- 3 From the Plane type list, choose Face parallel.
- **4** Click the **Comextents** button in the **Graphics** toolbar.

5 On the object rot1(1), select Boundary 4 only.



- 6 Click to expand the Local Coordinate System section. In the Rotation text field, type 180.
- 7 Click 🟢 Build All Objects.

The geometry with the coordinate system created by the work plane is depicted in Figure 3. The local coordinate system is used for orienting the applied forces and moment on the receiver.

Proceed to the **Definitions** node and add variables, create selections, integration operators, and add the **Mass Properties** node. The last is used to calculate the geometric center of the receiver box. The center of mass and other quantities are defined relative to the geometric center.

## DEFINITIONS

Main Variables

- I In the Model Builder window, expand the Component I (compl)>Definitions node.
- 2 Right-click Definitions and choose Variables.
- 3 In the Settings window for Variables, type Main Variables in the Label text field.
- 4 Locate the Variables section. Click 📂 Load from File.
- 5 Browse to the model's Application Libraries folder and double-click the file lumped\_receiver\_vibroacoustic\_variables\_main.txt.

## **Receiver Variables**

- I Right-click **Definitions** and choose **Variables**.
- 2 In the Settings window for Variables, type Receiver Variables in the Label text field.
- **3** Locate the Variables section. Click *b* Load from File.
- 4 Browse to the model's Application Libraries folder and double-click the file lumped\_receiver\_vibroacoustic\_variables\_receiver.txt.

#### Mass Properties I (mass I)

- I Right-click Definitions and choose Physics Utilities>Mass Properties.
- 2 In the Settings window for Mass Properties, locate the Source Selection section.
- **3** Click **Clear Selection**.
- **4** Click the  $\longleftrightarrow$  **Zoom Extents** button in the **Graphics** toolbar.
- **5** Select Domain 1 only.
- 6 Click the 🖂 Wireframe Rendering button in the Graphics toolbar.

#### Transducer

- I In the **Definitions** toolbar, click 🝡 **Explicit**.
- 2 In the Settings window for Explicit, type Transducer in the Label text field.
- **3** Select Domains 1, 2, and 5 only.

## Inner Tube

- I In the Definitions toolbar, click http://www.click.ic.
- 2 In the Settings window for Explicit, type Inner Tube in the Label text field.
- **3** Select Domains 10, 16, and 19 only.

#### Acoustic-Structure Interaction

- I In the **Definitions** toolbar, click http://www.click
- 2 In the Settings window for Explicit, type Acoustic-Structure Interaction in the Label text field.
- 3 Locate the Input Entities section. From the Geometric entity level list, choose Boundary.
- 4 Select Boundaries 35, 36, 40, and 43 only.

#### Inlet

- I In the **Definitions** toolbar, click **here Explicit**.
- 2 In the Settings window for Explicit, type Inlet in the Label text field.
- 3 Locate the Input Entities section. From the Geometric entity level list, choose Boundary.

4 Select Boundary 34 only.

## Microphone

- I In the **Definitions** toolbar, click **here Explicit**.
- 2 In the Settings window for Explicit, type Microphone in the Label text field.
- 3 Locate the Input Entities section. From the Geometric entity level list, choose Boundary.
- 4 Select Boundary 78 only.

## Silicone Tubing

- I In the **Definitions** toolbar, click **here Explicit**.
- 2 In the Settings window for Explicit, type Silicone Tubing in the Label text field.
- **3** Select Domains 3, 4, 6–9, 11, 12, 14, 15, 17, and 18 only.

## Solid Domains

- I In the **Definitions** toolbar, click 🛅 **Union**.
- 2 In the Settings window for Union, type Solid Domains in the Label text field.
- 3 Locate the Input Entities section. Under Selections to add, click + Add.
- 4 In the Add dialog box, in the Selections to add list, choose Transducer and Silicone Tubing.
- 5 Click OK.

## Air Domains

- I In the Definitions toolbar, click 🖣 Explicit.
- 2 In the Settings window for Explicit, type Air Domains in the Label text field.
- **3** Select Domains 10, 13, 16, and 19 only.

## Integration 1 (intop1)

- I In the Definitions toolbar, click 🖉 Nonlocal Couplings and choose Integration.
- 2 In the Settings window for Integration, type intop\_mic in the Operator name text field.
- **3** Locate the **Source Selection** section. From the **Geometric entity level** list, choose **Boundary**.
- 4 From the Selection list, choose Microphone.

Use the coordinates defined in **Work Plane 3** to define a base vector coordinate system to be used in the physics. Then proceed to setting up the materials.

Base Vector System 2 (sys2)

I In the Definitions toolbar, click  $\int_{-\infty}^{z}$  Coordinate Systems and choose Base Vector System.

- 2 In the Settings window for Base Vector System, locate the Relative to System from Geometry section.
- 3 From the Work plane list, choose Work Plane 3 (wp3).

#### DEFINITIONS

In the Model Builder window, collapse the Component I (compl)>Definitions node.

#### GEOMETRY I

In the Model Builder window, collapse the Component I (compl)>Geometry I node.

## ADD MATERIAL

- I In the Home toolbar, click 👯 Add Material to open the Add Material window.
- **2** Go to the **Add Material** window.
- 3 In the tree, select Built-in>Air.
- 4 Click Add to Component in the window toolbar.

## MATERIALS

Air (mat1)

- I In the Settings window for Material, locate the Geometric Entity Selection section.
- 2 From the Selection list, choose Air Domains.

#### Tubing (Silicone)

- I In the Model Builder window, right-click Materials and choose Blank Material.
- 2 In the Settings window for Material, type Tubing (Silicone) in the Label text field.
- **3** Locate the **Geometric Entity Selection** section. From the **Selection** list, choose **Silicone Tubing**.
- **4** Locate the **Material Contents** section. In the table, enter the following settings:

Property	Variable	Value	Unit	Property group
Young's modulus	E	7e6[Pa]	Pa	Young's modulus and Poisson's ratio
Poisson's ratio	nu	0.47	1	Young's modulus and Poisson's ratio
Density	rho	1100[kg/m^3]	kg/m³	Basic

5 In the Home toolbar, click 🙀 Add Material to close the Add Material window.

Proceed to setting up the physics. First, add the selections for the two domain physics (acoustics and solid mechanics) and note that the warning in materials node disappears. Then proceed to the detailed physics setup.

## SOLID MECHANICS (SOLID)

- I In the Settings window for Solid Mechanics, locate the Domain Selection section.
- 2 From the Selection list, choose Solid Domains.

## PRESSURE ACOUSTICS, FREQUENCY DOMAIN (ACPR)

- In the Model Builder window, under Component I (compl) click Pressure Acoustics, Frequency Domain (acpr).
- 2 In the Settings window for Pressure Acoustics, Frequency Domain, locate the Domain Selection section.
- 3 From the Selection list, choose Air Domains.

In the **Electrical Circuit** physics, set up the lumped model of the Knowles TEC-30033 receiver by loading its SPICE circuit. Then set up the voltage source driving the receiver and the external coupling to the acoustics, that is, the pressure at the outlet of the miniature loudspeaker.

## ELECTRICAL CIRCUIT (CIR)

Subcircuit Definition TEC30033 (TEC30033)

- I In the Model Builder window, under Component I (compl) right-click Electrical Circuit (cir) and choose Import SPICE Netlist.
- 2 Browse to the model's Application Libraries folder and double-click the file lumped\_receiver\_vibroacoustic\_TEC30033.cir.

#### Resistor RKARM (RKARM)

- I In the Model Builder window, expand the Subcircuit Definition TEC30033 (TEC30033) node, then click Resistor RKARM (RKARM).
- 2 In the Settings window for Resistor, locate the Device Parameters section.
- **3** In the *R* text field, type GKarm.

Subcircuit Definition TEC30033 (TEC30033)

In the Model Builder window, collapse the Subcircuit Definition TEC30033 (TEC30033) node.

Subcircuit Instance 1 (X1)

I In the Electrical Circuit toolbar, click 🗰 Subcircuit Instance.

2 In the Settings window for Subcircuit Instance, locate the Node Connections section.

3 From the Name of subcircuit link list, choose Subcircuit Definition TEC30033 (TEC30033).

**4** In the table, enter the following settings:

Local node names	Node names
PI	p1
NI	0
P2	p2
N2	0

Voltage Source 1 (V1)

I In the Electrical Circuit toolbar, click 🔅 Voltage Source.

2 In the Settings window for Voltage Source, locate the Node Connections section.

**3** In the table, enter the following settings:

Label	Node names
Ρ	p1
n	0

**4** Locate the **Device Parameters** section. In the  $v_{\rm src}$  text field, type V0.

External I vs. U I (IvsUI)

I In the Electrical Circuit toolbar, click - External I vs. U.

Set up the external source that couples to the Lumped Port in pressure acoustics.

2 In the Settings window for External I vs. U, locate the Node Connections section.

**3** In the table, enter the following settings:

Label	Node names
Ρ	p2
n	0

Now, set up the **Solid Mechanics** physics. Add damping to the silicone earmold tube (and define its material property) and then proceed to setting up the **Rigid Material** properties. The transducer is modeled through rigid body motion with given center of mass and moment of inertia. The motion and vibration characteristics are given by coupling the lumped spice model to the rigid domain by applying forces and moment.

## SOLID MECHANICS (SOLID)

## Linear Elastic Material I

In the Model Builder window, under Component I (comp1)>Solid Mechanics (solid) click Linear Elastic Material I.

#### Damping I

- I In the Physics toolbar, click 📃 Attributes and choose Damping.
- 2 In the Settings window for Damping, locate the Damping Settings section.
- 3 From the Damping type list, choose Isotropic loss factor.

## MATERIALS

Tubing (Silicone) (mat2)

- I In the Model Builder window, under Component I (compl)>Materials click Tubing (Silicone) (mat2).
- 2 In the Settings window for Material, locate the Material Contents section.
- **3** In the table, enter the following settings:

Property	Variable	Value	Unit	Property group
Isotropic structural loss	eta_s	0.1	I	Basic
factor				

## SOLID MECHANICS (SOLID)

Rigid Material I

- I In the Physics toolbar, click 🔚 Domains and choose Rigid Material.
- 2 In the Settings window for Rigid Material, locate the Domain Selection section.
- 3 From the Selection list, choose Transducer.
- **5** Specify the **X**<sub>c</sub> vector as

CMx	x
СМу	у
CMz	z

Mass and Moment of Inertia 1

I In the Physics toolbar, click 🔙 Attributes and choose Mass and Moment of Inertia.

- 2 In the Settings window for Mass and Moment of Inertia, locate the Coordinate System Selection section.
- 3 From the Coordinate system list, choose Base Vector System 2 (sys2).
- 4 Locate the Center of Mass section. From the list, choose User defined.
- **5** Specify the  $\mathbf{X}_m$  vector as

CMx	х
СМу	у
CMz	z

6 Locate the Mass and Moment of Inertia section. In the *m* text field, type Mass.

7 From the list, choose Symmetric.

8 In the I table, enter the following settings:

Ixx	Ixy	Ixz
lxy	Іуу	Iyz
lxz	lyz	Izz

Rigid Material I

In the Model Builder window, click Rigid Material I.

Applied Force 1

- I In the Physics toolbar, click 🦳 Attributes and choose Applied Force.
- 2 In the Settings window for Applied Force, locate the Coordinate System Selection section.
- 3 From the Coordinate system list, choose Base Vector System 2 (sys2).
- 4 Locate the Location section. From the list, choose User defined.
- **5** Specify the **X**<sub>*p*</sub> vector as

CMx x CMy y CMz z

6 Locate the Applied Force section. Specify the F vector as

Fx	хI
0	x2
Fz	x3

## Rigid Material I

In the Model Builder window, click Rigid Material I.

#### Applied Moment I

- I In the Physics toolbar, click 📃 Attributes and choose Applied Moment.
- **2** In the **Settings** window for **Applied Moment**, locate the **Coordinate System Selection** section.
- 3 From the Coordinate system list, choose Base Vector System 2 (sys2).
- 4 Locate the Applied Moment section. Specify the M vector as

0	хI
Му	x2
0	x3

Fixed Constraint 1

- I In the Physics toolbar, click 🔚 Boundaries and choose Fixed Constraint.
- **2** Select Boundaries 55, 56, 60, and 63 only.

Set up the acoustics model. The **Narrow Region Acoustics** feature is used to model the thermoviscous losses in the narrow earmold tube. A simple RCL impedance condition could have been used to model the mechanical properties of the microphone located at the end of the 2 cc coupler. This is omitted here and the microphone is assumed rigid.

## PRESSURE ACOUSTICS, FREQUENCY DOMAIN (ACPR)

In the Model Builder window, under Component I (compl) click Pressure Acoustics, Frequency Domain (acpr).

Lumped Port I

I In the Physics toolbar, click 🔚 Boundaries and choose Lumped Port.

The **Lumped Port** has built-in functionality that couples the port boundary to the Electric Circuit physics.

- 2 Select Boundary 34 only.
- 3 In the Settings window for Lumped Port, locate the Lumped Port Properties section.
- 4 From the Connection type list, choose Circuit.

Now, finalize the coupling between the port and the circuit.

## ELECTRICAL CIRCUIT (CIR)

## External I vs. U I (IvsUI)

- I In the Model Builder window, under Component I (comp1)>Electrical Circuit (cir) click External I vs. U I (IvsU1).
- 2 In the Settings window for External I vs. U, locate the External Device section.
- **3** From the V list, choose **Voltage from lumped port (acpr/lport1)**.

## PRESSURE ACOUSTICS, FREQUENCY DOMAIN (ACPR)

In the Model Builder window, under Component I (compl) click Pressure Acoustics, Frequency Domain (acpr).

Narrow Region Acoustics 1

- I In the Physics toolbar, click 🔚 Domains and choose Narrow Region Acoustics.
- 2 In the Settings window for Narrow Region Acoustics, locate the Domain Selection section.
- **3** From the **Selection** list, choose **Inner Tube**.
- 4 Locate the Duct Properties section. From the Duct type list, choose Circular duct.
- **5** In the *a* text field, type Td/2.

Finally, set up the multiphysics coupling between acoustics and structure. Then proceed to meshing.

## MULTIPHYSICS

Acoustic-Structure Boundary I (asb1)

- I In the Physics toolbar, click A Multiphysics Couplings and choose Boundary>Acoustic-Structure Boundary.
- **2** In the **Settings** window for **Acoustic-Structure Boundary**, locate the **Boundary Selection** section.
- **3** From the Selection list, choose Acoustic-Structure Interaction.

#### MESH I

In this model, the mesh is set up manually. Proceed by directly adding the desired mesh components. Use a swept mesh through the solids to make sure that there are at least elements through the thickness. This is done to make sure that the bending stiffness is correctly captured.

#### Mapped I

I In the Mesh toolbar, click  $\bigwedge$  Boundary and choose Mapped.

**2** Select Boundaries 10, 11, 17, 20, and 24 only.

## Distribution I

- I Right-click Mapped I and choose Distribution.
- 2 Select Edges 24, 32, and 33 only.
- 3 In the Settings window for Distribution, locate the Distribution section.
- 4 In the Number of elements text field, type 3.

#### Size

- I In the Model Builder window, under Component I (compl)>Mesh I click Size.
- 2 In the Settings window for Size, locate the Element Size section.
- 3 Click the **Custom** button.
- **4** Locate the **Element Size Parameters** section. In the **Maximum element size** text field, type 2\*Td.
- 5 In the Minimum element size text field, type Td/2.

### Swept I

- I In the Mesh toolbar, click A Swept.
- 2 In the Settings window for Swept, locate the Domain Selection section.
- 3 From the Geometric entity level list, choose Domain.
- 4 Select Domains 2–12 and 14–19 only.

## Size I

- I Right-click Swept I and choose Size.
- 2 In the Settings window for Size, locate the Element Size section.
- 3 Click the **Custom** button.
- 4 Locate the Element Size Parameters section.
- 5 Select the Maximum element size check box. In the associated text field, type Td/2.

## Free Tetrahedral I

I In the Mesh toolbar, click \land Free Tetrahedral.

2 In the Model Builder window, right-click Mesh I and choose Build All. The mesh should look like this.



## STUDY I

## Step 1: Frequency Domain

Some manual setup of the solver is necessary. The default for the current combination of physics is to use a segregated solution approach. In this model, it is necessary to use a fully coupled solver. Generate the default solver and then make a small change. The model is solved from 10 Hz to 10 kHz in steps of 1/12 octaves.

- I In the Model Builder window, under Study I click Step I: Frequency Domain.
- 2 In the Settings window for Frequency Domain, locate the Study Settings section.
- 3 Click Range.
- 4 In the Range dialog box, choose ISO preferred frequencies from the Entry method list.
- **5** In the **Start frequency** text field, type **10**.
- 6 In the **Stop frequency** text field, type fmax.
- 7 From the Interval list, choose 1/12 octave.
- 8 Click Replace.

**9** In the **Home** toolbar, click **= Compute**.

## RESULTS

## Displacement (solid)

Proceed to postprocessing the results. First, use the default plots and make some modifications to generate Figure 10, Figure 11, and Figure 12. If you zoom in on the **Sound Pressure Level (acpr)** plot you can see the location of the geometric center and the center of mass (added using the **Annotation** option). They are located close together. Change the frequency parameter to look at the solution for one of the solved frequencies.

Secondly, proceed to plotting the acoustic and vibration response and compare it with measurement data. This will recreate Figure 6, Figure 7, Figure 8, and Figure 9. The measurement data is imported as interpolation functions under the **Definitions** node.

I In the Settings window for 3D Plot Group, type Displacement (solid) in the Label text field.

## Volume 1

- I In the Model Builder window, expand the Displacement (solid) node, then click Volume I.
- 2 In the Settings window for Volume, locate the Expression section.
- 3 In the **Expression** text field, type solid.disp.
- **4** In the **Displacement (solid)** toolbar, click **I** Plot.

#### Sound Pressure Level (acpr)

- I In the Model Builder window, under Results click Sound Pressure Level (acpr).
- 2 In the Sound Pressure Level (acpr) toolbar, click 🗿 Plot.

#### Annotation I

- I Right-click Sound Pressure Level (acpr) and choose Annotation.
- 2 In the Settings window for Annotation, locate the Annotation section.
- 3 In the Text text field, type Center of mass.
- **4** Locate the **Position** section. In the **x** text field, type CMx.
- **5** In the **y** text field, type CMy.
- 6 In the z text field, type CMz.

#### Annotation 2

- I Right-click Sound Pressure Level (acpr) and choose Annotation.
- 2 In the Settings window for Annotation, locate the Annotation section.

- 3 In the Text text field, type Geometric center of box.
- 4 Locate the **Position** section. In the **x** text field, type mass1.CMX.
- **5** In the **y** text field, type mass1.CMY.
- 6 In the z text field, type mass1.CMZ.
- 7 Locate the Coloring and Style section. From the Color list, choose Red.
- 8 From the Anchor point list, choose Lower left.
- 9 In the Sound Pressure Level (acpr) toolbar, click 🗿 Plot.

#### Isosurface 1

- I In the Model Builder window, expand the Acoustic Pressure, Isosurfaces (acpr) node, then click Isosurface 1.
- 2 In the Settings window for Isosurface, locate the Levels section.
- 3 In the Total levels text field, type 20.

#### Acoustic Pressure, Isosurfaces (acpr)

- I In the Model Builder window, click Acoustic Pressure, Isosurfaces (acpr).
- 2 In the Settings window for 3D Plot Group, click to expand the Title section.
- 3 From the Title type list, choose Label.

#### Coordinate System Volume 1

- I In the Acoustic Pressure, Isosurfaces (acpr) toolbar, click i More Plots and choose Coordinate System Volume.
- **2** In the **Settings** window for **Coordinate System Volume**, locate the **Coordinate System** section.
- 3 From the Coordinate system list, choose Base Vector System 2 (sys2).
- 4 Locate the **Positioning** section. Find the **x grid points** subsection. From the **Entry method** list, choose **Coordinates**.
- 5 In the Coordinates text field, type CMx.
- 6 Find the y grid points subsection. From the Entry method list, choose Coordinates.
- 7 In the **Coordinates** text field, type CMy.
- 8 Find the z grid points subsection. From the Entry method list, choose Coordinates.
- 9 In the Coordinates text field, type CMz.
- **IO** In the Acoustic Pressure, Isosurfaces (acpr) toolbar, click **ID** Plot.

#### **GLOBAL DEFINITIONS**

Interpolation 1 (int1)

- I In the Home toolbar, click f(X) Functions and choose Global>Interpolation.
- 2 In the Settings window for Interpolation, locate the Definition section.
- 3 From the Data source list, choose File.
- 4 Click 📂 Browse.
- 5 Browse to the model's Application Libraries folder and double-click the file lumped\_receiver\_vibroacoustic\_SPL\_data.txt.
- 6 Click III- Import.
- 7 Find the Functions subsection. In the table, enter the following settings:

Function name	Position in file	
SPL_data	1	

Interpolation 2 (int2)

- I In the Home toolbar, click f(X) Functions and choose Global>Interpolation.
- 2 In the Settings window for Interpolation, locate the Definition section.
- 3 From the Data source list, choose File.
- 4 Click 📂 Browse.
- 5 Browse to the model's Application Libraries folder and double-click the file lumped\_receiver\_vibroacoustic\_xvel\_data\_01.txt.
- 6 In the Number of arguments text field, type 1.
- 7 Click **[I** Import.]
- 8 Find the Functions subsection. In the table, enter the following settings:

Function name	Position in file	
xvel_real_01	1	
xvel_imag_01	2	

Interpolation 3 (int3)

- I In the Home toolbar, click f(X) Functions and choose Global>Interpolation.
- 2 In the Settings window for Interpolation, locate the Definition section.
- 3 From the Data source list, choose File.
- 4 Click 📂 Browse.

- 5 Browse to the model's Application Libraries folder and double-click the file lumped\_receiver\_vibroacoustic\_xvel\_data\_02.txt.
- 6 Click **[F** Import.
- 7 In the Function name text field, type xvel\_dB\_02.

## Interpolation 4 (int4)

- I In the Home toolbar, click f(X) Functions and choose Global>Interpolation.
- 2 In the Settings window for Interpolation, locate the Definition section.
- 3 From the Data source list, choose File.
- 4 Click 📂 Browse.
- 5 Browse to the model's Application Libraries folder and double-click the file lumped\_receiver\_vibroacoustic\_zvel\_data\_01.txt.
- 6 In the Number of arguments text field, type 1.
- 7 Click **[I** Import.]
- 8 Find the Functions subsection. In the table, enter the following settings:

Function name	Position in file
zvel_real_01	1
zvel_imag_01	2

Interpolation 5 (int5)

- I In the Home toolbar, click f(X) Functions and choose Global>Interpolation.
- 2 In the Settings window for Interpolation, locate the Definition section.
- 3 From the Data source list, choose File.
- 4 Click 📂 Browse.
- 5 Browse to the model's Application Libraries folder and double-click the file lumped\_receiver\_vibroacoustic\_zvel\_data\_02.txt.
- 6 Click III- Import.
- 7 In the Function name text field, type zvel\_dB\_02.

## RESULTS

Grid ID I

- I In the **Results** toolbar, click **More Datasets** and choose **Grid>Grid ID**.
- 2 In the Settings window for Grid ID, locate the Data section.
- **3** From the **Source** list, choose **Function**.

- 4 From the Function list, choose All.
- 5 Locate the Parameter Bounds section. In the Name text field, type f.
- **6** In the **Minimum** text field, type 100.
- 7 In the Maximum text field, type 10000.

#### SPL at Microphone

- I In the Results toolbar, click  $\sim$  ID Plot Group.
- 2 In the Settings window for ID Plot Group, type SPL at Microphone in the Label text field.
- 3 Click to expand the Title section. From the Title type list, choose Label.
- 4 Locate the **Plot Settings** section.
- 5 Select the y-axis label check box. In the associated text field, type Level (dB SPL).
- 6 Locate the Axis section. Select the x-axis log scale check box.
- 7 Locate the Legend section. From the Position list, choose Lower left.

## Global I

- I Right-click SPL at Microphone and choose Global.
- 2 In the Settings window for Global, locate the y-Axis Data section.
- **3** In the table, enter the following settings:

Expression	Unit	Description
20*log10(abs(pmic/V0/acpr.pref_SPL))		Simulation

## Line Graph 1

- I In the Model Builder window, right-click SPL at Microphone and choose Line Graph.
- 2 In the Settings window for Line Graph, locate the Data section.
- 3 From the Dataset list, choose Grid ID I.
- 4 Locate the y-Axis Data section. In the Expression text field, type SPL\_data(f).
- 5 Locate the x-Axis Data section. From the Parameter list, choose Expression.
- 6 In the Expression text field, type f[Hz/m].
- 7 Select the **Description** check box. In the associated text field, type freq.
- 8 Click to expand the Legends section. Select the Show legends check box.
- 9 From the Legends list, choose Manual.

**IO** In the table, enter the following settings:

## Legends

#### Measurements

II In the SPL at Microphone toolbar, click 💿 Plot.

## x-Velocity

- I In the Home toolbar, click 🚛 Add Plot Group and choose ID Plot Group.
- 2 In the Settings window for ID Plot Group, type x-Velocity in the Label text field.
- **3** Locate the **Title** section. From the **Title type** list, choose **Label**.
- 4 Locate the **Plot Settings** section.
- 5 Select the y-axis label check box. In the associated text field, type x-velocity (dB rel. 1 m/s/V).
- 6 Locate the Axis section. Select the x-axis log scale check box.
- 7 Locate the Legend section. From the Position list, choose Lower right.

### Point Graph 1

- I Right-click x-Velocity and choose Point Graph.
- **2** Select Points 1 and 2 only.
- 3 In the Settings window for Point Graph, locate the y-Axis Data section.
- **4** In the **Expression** text field, type 20\*log10(abs(vx\_local)/V0).
- 5 Click to expand the Legends section. Select the Show legends check box.
- 6 Find the Prefix and suffix subsection. In the Prefix text field, type Point .
- 7 In the Suffix text field, type , Simulation.

#### Line Graph I

- I In the Model Builder window, right-click x-Velocity and choose Line Graph.
- 2 In the Settings window for Line Graph, locate the Data section.
- 3 From the Dataset list, choose Grid ID I.
- 4 Locate the y-Axis Data section. In the Expression text field, type 20\* log10(sqrt(xvel\_real\_01(f)^2+xvel\_imag\_01(f)^2)).
- 5 Locate the x-Axis Data section. From the Parameter list, choose Expression.
- 6 In the **Expression** text field, type f[Hz/m].
- 7 Select the **Description** check box. In the associated text field, type freq.
- 8 Locate the Legends section. Select the Show legends check box.

9 From the Legends list, choose Manual.

**IO** In the table, enter the following settings:

Legends

Measurements (series 1)

Line Graph 2

- I Right-click Line Graph I and choose Duplicate.
- 2 In the Settings window for Line Graph, locate the y-Axis Data section.
- **3** In the **Expression** text field, type xvel\_dB\_02(f).
- 4 Locate the Legends section. In the table, enter the following settings:

#### Legends

Measurements (series 2)

5 In the x-Velocity toolbar, click **I** Plot.

y-Velocity

- I In the Home toolbar, click 🚛 Add Plot Group and choose ID Plot Group.
- 2 In the Settings window for ID Plot Group, type y-Velocity in the Label text field.
- 3 Locate the Title section. From the Title type list, choose Label.
- 4 Locate the **Plot Settings** section.
- 5 Select the y-axis label check box. In the associated text field, type y-velocity (dB rel. 1 m/s/V).
- 6 Locate the Axis section. Select the x-axis log scale check box.

Point Graph 1

- I Right-click y-Velocity and choose Point Graph.
- **2** Select Point 2 only.
- 3 In the Settings window for Point Graph, locate the y-Axis Data section.
- 4 In the Expression text field, type 20\*log10(abs(vy\_local)/V0).
- **5** In the **y-Velocity** toolbar, click **I** Plot.

#### z-Velocity

- I In the Home toolbar, click 🚛 Add Plot Group and choose ID Plot Group.
- 2 In the Settings window for ID Plot Group, type z-Velocity in the Label text field.
- 3 Locate the Title section. From the Title type list, choose Label.

- 4 Locate the **Plot Settings** section.
- 5 Select the y-axis label check box. In the associated text field, type z-velocity (dB rel. 1 m/s/V).
- 6 Locate the Axis section. Select the x-axis log scale check box.
- 7 Locate the Legend section. From the Position list, choose Lower right.

#### Point Graph 1

- I Right-click z-Velocity and choose Point Graph.
- **2** Select Points 17 and 18 only.
- 3 In the Settings window for Point Graph, locate the y-Axis Data section.
- **4** In the **Expression** text field, type 20\*log10(abs(vz\_local)/V0).
- 5 Locate the Legends section. Select the Show legends check box.
- 6 Find the Prefix and suffix subsection. In the Prefix text field, type Point .
- 7 In the Suffix text field, type , Simulation.

#### Line Graph 1

- I In the Model Builder window, right-click z-Velocity and choose Line Graph.
- 2 In the Settings window for Line Graph, locate the Data section.
- 3 From the Dataset list, choose Grid ID I.
- 4 Locate the y-Axis Data section. In the Expression text field, type 20\* log10(sqrt(zvel\_real\_01(f)^2+zvel\_imag\_01(f)^2)).
- 5 Locate the x-Axis Data section. From the Parameter list, choose Expression.
- **6** In the **Expression** text field, type f[Hz/m].
- 7 Select the **Description** check box. In the associated text field, type freq.
- 8 Locate the Legends section. Select the Show legends check box.
- 9 From the Legends list, choose Manual.

**IO** In the table, enter the following settings:

## Legends

Measurements (series 1)

Line Graph 2

- I Right-click Line Graph I and choose Duplicate.
- 2 In the Settings window for Line Graph, locate the y-Axis Data section.
- **3** In the **Expression** text field, type zvel\_dB\_02(f).

4 Locate the Legends section. In the table, enter the following settings:

# Legends Measurements (series 2)

**5** In the **z-Velocity** toolbar, click **I** Plot.

# Geometry Modeling Instructions

From the File menu, choose New.

## NEW

In the New window, click Slank Model.

## GLOBAL DEFINITIONS

### Parameters 1

- I In the Model Builder window, under Global Definitions click Parameters I.
- 2 In the Settings window for Parameters, locate the Parameters section.
- 3 Click 📂 Load from File.
- **4** Browse to the model's Application Libraries folder and double-click the file lumped\_receiver\_vibroacoustic\_geom\_sequence\_parameters.txt.

## ADD COMPONENT

In the Home toolbar, click 🚫 Add Component and choose **3D**.

## GEOMETRY I

Block I (blk1)

- I In the **Geometry** toolbar, click 🗍 **Block**.
- 2 In the Settings window for Block, locate the Size and Shape section.
- 3 In the Width text field, type Lx.
- **4** In the **Depth** text field, type Ly.
- **5** In the **Height** text field, type Lz.
- 6 Locate the Position section. From the Base list, choose Center.

### Cylinder I (cyl1)

I In the **Geometry** toolbar, click 💭 **Cylinder**.

- 2 In the Settings window for Cylinder, locate the Size and Shape section.
- 3 In the Radius text field, type Td/2.
- **4** In the **Height** text field, type TL.
- **5** Locate the **Position** section. In the **x** text field, type Lx/2.
- 6 In the z text field, type Th.
- 7 Locate the Axis section. From the Axis type list, choose x-axis.

Cylinder 2 (cyl2)

- I In the Geometry toolbar, click 💭 Cylinder.
- 2 In the Settings window for Cylinder, locate the Size and Shape section.
- 3 In the Radius text field, type Td/2+Ttube.
- 4 In the **Height** text field, type Ltube.
- 5 Locate the **Position** section. In the **x** text field, type Lx/2+SToffset.
- 6 In the z text field, type Th.
- 7 Locate the Axis section. From the Axis type list, choose x-axis.
- 8 Click to expand the Layers section. In the table, enter the following settings:

Layer name	Thickness (m)
Layer 1	Ttube

Cylinder 3 (cyl3)

- I In the **Geometry** toolbar, click 🔲 **Cylinder**.
- 2 In the Settings window for Cylinder, locate the Size and Shape section.
- **3** In the **Radius** text field, type Td/2.
- 4 In the **Height** text field, type LtubeC.
- 5 Locate the Position section. In the x text field, type Lx/2+SToffset+Ltube-(LtubeC-CToffset).
- 6 In the z text field, type Th.
- 7 Locate the Axis section. From the Axis type list, choose x-axis.

Cylinder 4 (cyl4)

- I In the **Geometry** toolbar, click **(\_\_\_\_) Cylinder**.
- 2 In the Settings window for Cylinder, locate the Size and Shape section.
- 3 In the Radius text field, type d\_2cc/2.
- 4 In the **Height** text field, type L\_2cc.

- 5 Locate the Position section. In the x text field, type Lx/2+SToffset+Ltube+CToffset.
- 6 In the z text field, type Th.
- 7 Locate the Axis section. From the Axis type list, choose x-axis.

## Work Plane I (wp1)

- I In the Geometry toolbar, click 🖶 Work Plane.
- 2 In the Settings window for Work Plane, locate the Plane Definition section.
- 3 From the Plane type list, choose Face parallel.
- 4 Click the 🕀 Wireframe Rendering button in the Graphics toolbar.
- 5 On the object cyll, select Boundary 4 only.

## Work Plane 2 (wp2)

- I In the Geometry toolbar, click 📥 Work Plane.
- 2 In the Settings window for Work Plane, locate the Plane Definition section.
- 3 From the Plane type list, choose Face parallel.
- 4 On the object cyl3, select Boundary 3 only.

Partition Objects 1 (parl)

- I In the Geometry toolbar, click 📕 Booleans and Partitions and choose Partition Objects.
- 2 Select the object cyl2 only.
- 3 In the Settings window for Partition Objects, locate the Partition Objects section.
- 4 From the **Partition with** list, choose **Work plane**.
- 5 From the Work plane list, choose Work Plane I (wpl).
- 6 Click 틤 Build Selected.

#### Partition Objects 2 (par2)

- I In the Geometry toolbar, click provide Booleans and Partitions and choose Partition Objects.
- 2 Select the object parl only.
- 3 In the Settings window for Partition Objects, locate the Partition Objects section.
- 4 From the Partition with list, choose Work plane.
- 5 Click 틤 Build Selected.

## Move I (movI)

- I In the Geometry toolbar, click 💭 Transforms and choose Move.
- 2 Click in the Graphics window and then press Ctrl+A to select all objects.
- 3 In the Settings window for Move, locate the Displacement section.

- 4 In the x text field, type 1 [mm].
- 5 In the y text field, type 1 [mm].
- 6 Click 틤 Build Selected.

Rotate | (rot |)

- I In the Geometry toolbar, click [7] Transforms and choose Rotate.
- 2 Click in the Graphics window and then press Ctrl+A to select all objects.
- 3 In the Settings window for Rotate, locate the Rotation section.
- 4 In the Angle text field, type 30.
- 5 Locate the Point on Axis of Rotation section. In the x text field, type 1[mm].
- 6 In the y text field, type 1 [mm].
- 7 Locate the Rotation section. From the Axis type list, choose y-axis.
- 8 Click 🟢 Build All Objects.

## Form Union (fin)

- I In the Geometry toolbar, click 📗 Build All.
- 2 Click the **Comextents** button in the **Graphics** toolbar.

The finalized geometry should look like the figure below.



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