

Eigenmodes in a Muffler with Elastic Walls

Introduction

The model below is an extension of the model Eigenmodes in a Muffler. The latter studied the propagation of acoustic waves through the cross section of the Absorptive Muffler chamber, provided that the muffler walls were sound hard. Here the muffler walls are considered to be made of a linear elastic material to account for their influence on the modes propagating through the cross section of the chamber. This is done using the Acoustic-Structure Interaction multiphysics interface.

Model Definition

The muffler chamber has the same shape and dimensions as in example (Eigenmodes in a Muffler). The only difference is that an elastic wall of the thickness W surrounds the inner (acoustic) domain as shown in Figure 1.



Figure 1: Geometry of model: the chamber of the same cross section as in Eigenmodes in a Muffler with the elastic wall to the thickness W.

The chamber is filled with air and the elastic wall is made of steel. Assuming the timeharmonic process with the angular frequency $\omega = 2\pi f$, the Mode Analysis study solves the equations for the out-of-plane wave number κ_z as the function of given parameter *f*. The cutoff frequency for the j-th mode is then calculated as

$$f_j = \frac{\sqrt{\omega^2 - c^2 \kappa_z^2}}{2\pi} \tag{1}$$

Results and Discussion

The results of the simulations show that the presence of a thin elastic wall significantly affects the number of modes propagating through the cross section of the chamber. It becomes greater than that for the chamber with the sound hard wall boundary studied in the model Eigenmodes in a Muffler. For example, the first propagating mode with the wave number lower than that of the plane wave mode has its cutoff frequency around 218 Hz. The former mode is shown in Figure 2; the latter, in Figure 3. Both figures shows the acoustic pressure in the chamber and the deformation of the muffler wall.



Figure 2: The plane wave propagating through the cross section of the chamber.



Figure 3: The first propagating mode different from the plane wave.

The model Eigenmodes in a Muffler gives the first least nonzero cutoff frequency equal to 635 Hz. A good understanding of the difference between these two cases is provided by the dispersion diagram shown in Figure 4, where the results corresponding to the presence of the wall are in black, and those obtained with the sound hard wall boundary condition are in red.

Another feature that distinguishes the chamber with the wall is that there can be modes having the wave number greater than that for the plane-wave mode, which is due to acoustics-structure interaction. In Figure 5 at a frequency of 400 Hz the pure acoustic plane wave-number is 7.33 1/m (the actual plane wave mode is at 7.27 1/m). While the wave number selected has a value of 7.87 1/m. This corresponds to the point above the red dotted line at 400 Hz in Figure 4. This means that their cutoff frequencies f_j calculated from Equation 1 become pure imaginary. Such modes do not exist in the model Eigenmodes in a Muffler.



Figure 4: Dispersion diagram: the red asterisks correspond to the sound hard wall boundary; the black circles, to the elastic wall.



Figure 5: A propagating mode with the wave number greater that of the plane wave.

Application Library path: Acoustics_Module/Automotive/ eigenmodes_in_muffler_elastic

Modeling Instructions

From the File menu, choose New.

NEW

In the New window, click **Model Wizard**.

MODEL WIZARD

- I In the Model Wizard window, click 9 2D.
- 2 In the Select Physics tree, select Acoustics>Acoustic-Structure Interaction>Acoustic-Solid Interaction, Frequency Domain.
- 3 Click Add.

- 4 Click \bigcirc Study.
- 5 In the Select Study tree, select Preset Studies for Selected Physics Interfaces> Mode Analysis.
- 6 Click 🗹 Done.

GLOBAL DEFINITIONS

Parameters 1

- I In the Model Builder window, under Global Definitions click Parameters I.
- 2 In the Settings window for Parameters, locate the Parameters section.
- **3** In the table, enter the following settings:

Name	Expression	Value	Description
W	1.5[mm]	0.0015 m	Muffler wall thickness
fO	500[Hz]	500 Hz	Cutoff frequency
c0	343[m/s]	343 m/s	Speed of sound in the air
lam0	c0/f0	0.686 m	Plane wave wavelength
k0	2*pi/lamO	9.1592 l/m	Free space plane-wave wave number

The parameter k_0 stands for the wave number of the plane wave generated at the cutoff frequency f_0 : $k_0 = 2\pi/\lambda_0 = 2\pi f_0/c_0$.

GEOMETRY I

- I In the Model Builder window, expand the Component I (compl)>Geometry I node, then click Geometry I.
- 2 In the Settings window for Geometry, locate the Units section.
- 3 From the Length unit list, choose mm.

Rectangle 1 (r1)

- I In the Geometry toolbar, click Rectangle.
- 2 In the Settings window for Rectangle, locate the Size and Shape section.
- 3 In the Width text field, type 150.
- 4 In the Height text field, type 150 + 2*W.
- 5 Locate the Position section. In the x text field, type -75.
- 6 In the y text field, type -75 W.

7 Click to expand the Layers section. In the table, enter the following settings:

Layer name	Thickness (mm)
Layer 1	W

8 Select the Layers on top check box.

9 Click 📄 Build Selected.

Circle I (c1)

- I In the **Geometry** toolbar, click 🕑 **Circle**.
- 2 In the Settings window for Circle, locate the Size and Shape section.
- **3** In the **Radius** text field, type **75** + W.
- 4 In the Sector angle text field, type 180.
- **5** Locate the **Position** section. In the **x** text field, type -75.
- 6 Locate the Rotation Angle section. In the Rotation text field, type 90.
- 7 Click to expand the Layers section. In the table, enter the following settings:

Layer name	Thickness (mm)	
Layer 1	W	

8 Click 틤 Build Selected.

Circle 2 (c2)

- I In the **Geometry** toolbar, click \bigcirc **Circle**.
- 2 In the Settings window for Circle, locate the Size and Shape section.
- 3 In the Radius text field, type 75 + W.
- 4 In the Sector angle text field, type 180.
- 5 Locate the **Position** section. In the **x** text field, type **75**.
- 6 Locate the Rotation Angle section. In the Rotation text field, type -90.
- 7 Locate the Layers section. In the table, enter the following settings:

Layer name	Thickness (mm)	
Layer 1	W	

8 Click 📄 Build Selected.

9 In the Geometry toolbar, click 🟢 Build All.

IO Click the **Come Extents** button in the **Graphics** toolbar.

The geometry should look like the one in Figure 1.

ADD MATERIAL

- I In the Home toolbar, click 🙀 Add Material to open the Add Material window.
- 2 Go to the Add Material window.
- 3 In the tree, select Built-in>Air.
- 4 Click Add to Component in the window toolbar.
- 5 In the tree, select Built-in>Steel AISI 4340.
- 6 Click Add to Component in the window toolbar.
- 7 In the Home toolbar, click 🙀 Add Material to close the Add Material window.

MATERIALS

Steel AISI 4340 (mat2) Select Domains 1–3, 6, 7, and 9 only.

PRESSURE ACOUSTICS, FREQUENCY DOMAIN (ACPR)

- I In the Model Builder window, under Component I (compl) click Pressure Acoustics, Frequency Domain (acpr).
- 2 In the Settings window for Pressure Acoustics, Frequency Domain, locate the Domain Selection section.
- 3 From the Selection list, choose Manual.
- 4 Select Domains 4, 5, and 8 only.

SOLID MECHANICS (SOLID)

- I In the Model Builder window, under Component I (compl) click Solid Mechanics (solid).
- 2 In the Settings window for Solid Mechanics, locate the Domain Selection section.
- 3 From the Selection list, choose Manual.
- **4** Select Domains 1–3, 6, 7, and 9 only.

Enable the **Out-of-plane mode extension (time-harmonic)** to perform the **Mode Analysis** for elastic waves in the muffler wall domain.

5 Locate the 2D Approximation section. Select the Out-of-plane mode extension (timeharmonic) check box.

MESH I

In this model, the mesh is set up manually. Proceed by directly adding the desired mesh component.

Free Triangular 1

- I In the Mesh toolbar, click Kree Triangular.
- 2 In the Settings window for Free Triangular, locate the Domain Selection section.
- 3 From the Geometric entity level list, choose Domain.
- **4** Select Domains 4, 5, and 8 only.

Size I

- I Right-click Free Triangular I and choose Size.
- 2 In the Settings window for Size, locate the Element Size section.
- 3 From the Predefined list, choose Extra fine.

Create a **Mapped** mesh for the muffler wall domain and choose the **Distribution** depth of 3 elements.

Mapped I

I In the Mesh toolbar, click Mapped.

Use the **Zoom Box** functionality to investigate the boundary layer mesh.

Distribution I

- I Right-click Mapped I and choose Distribution.
- **2** Select Boundary 2 only.
- 3 In the Settings window for Distribution, locate the Distribution section.
- 4 In the Number of elements text field, type 3.



STUDY I

Step 1: Mode Analysis

- I In the Model Builder window, under Study I click Step I: Mode Analysis.
- 2 In the Settings window for Mode Analysis, locate the Study Settings section.
- 3 In the Mode analysis frequency text field, type f0.

Set the solver to search for the first 6 propagating modes with the wave numbers lower that k_0 .

4 Select the Desired number of modes check box.

Search for the modes below the plane wave mode.

- 5 Select the Search for modes around check box. In the associated text field, type 1.1*k0.
- 6 From the Mode search method around shift list, choose Smaller real part.
- **7** In the **Home** toolbar, click **= Compute**.

RESULTS

Acoustic Pressure (acpr)

Now, select the out-of-plane wave number with a value close to 5.18. This is the first non-plane mode.

- I In the Settings window for 2D Plot Group, locate the Data section.
- 2 From the Out-of-plane wave number list, choose 5.1756.

Surface 2

- I In the Model Builder window, right-click Acoustic Pressure (acpr) and choose Surface.
- 2 In the Settings window for Surface, locate the Expression section.
- 3 In the Expression text field, type solid.disp.
- **4** Locate the **Coloring and Style** section. Click **Change Color Table**.
- 5 In the Color Table dialog box, select Aurora>AuroraBorealis in the tree.
- 6 Click OK.

Deformation I

I Right-click Surface 2 and choose Deformation.

The results should look like the one depicted in Figure 3.

Dispersion Relation Calculation

The next steps are optional and are added to show you how to get the dispersion relation curves that relate the frequency and the wave number. The dispersion relations for the chamber with elastic and hard walls are calculated and compared below.

ADD STUDY

- I In the Home toolbar, click $\stackrel{\text{reg}}{\longrightarrow}$ Add Study to open the Add Study window.
- 2 Go to the Add Study window.
- 3 Find the Studies subsection. In the Select Study tree, select Preset Studies for Selected Physics Interfaces>Mode Analysis.
- 4 Click Add Study in the window toolbar.
- 5 In the Home toolbar, click 2 Add Study to close the Add Study window.

STUDY 2

Step 1: Mode Analysis

I In the Settings window for Mode Analysis, locate the Study Settings section.

- 2 In the Mode analysis frequency text field, type f0.
- 3 Select the Desired number of modes check box.
- 4 Select the Search for modes around check box. In the associated text field, type 1.1*k0.
- 5 From the Mode search method around shift list, choose Smaller real part.

Parametric Sweep

- I In the Study toolbar, click **Parametric Sweep**.
- 2 In the Settings window for Parametric Sweep, locate the Study Settings section.
- 3 Click + Add.
- 4 From the list in the Parameter name column, choose f0 (Cutoff frequency).
- 5 Click Range.
- 6 In the Range dialog box, choose Number of values from the Entry method list.
- 7 In the Start text field, type 100.
- 8 In the Stop text field, type 700.
- 9 In the Number of values text field, type 51.
- IO Click Replace.
- II In the Model Builder window, click Study 2.
- 12 In the Settings window for Study, locate the Study Settings section.
- **I3** Clear the **Generate default plots** check box.
- **I4** In the **Study** toolbar, click **Compute**.

ADD STUDY

- I In the Study toolbar, click $\stackrel{\text{res}}{\longrightarrow}$ Add Study to open the Add Study window.
- 2 Go to the Add Study window.
- 3 Find the Studies subsection. In the Select Study tree, select Preset Studies for Selected Physics Interfaces>Mode Analysis.
- 4 Click Add Study in the window toolbar.
- 5 In the Study toolbar, click \sim Add Study to close the Add Study window.

STUDY 3

Step 1: Mode Analysis

- I In the Settings window for Mode Analysis, locate the Study Settings section.
- 2 In the Mode analysis frequency text field, type f0.

- **3** Select the **Desired number of modes** check box.
- 4 Select the Search for modes around check box. In the associated text field, type 1.1*k0.
- 5 From the Mode search method around shift list, choose Smaller real part.

Now, disable the **Solid Mechanics** physics which results in the outer wall of the acoustic domain to be sound hard.

6 Locate the Physics and Variables Selection section. In the table, clear the Solve for check box for Solid Mechanics (solid).

Parametric Sweep

- I In the Study toolbar, click **Parametric Sweep**.
- 2 In the Settings window for Parametric Sweep, locate the Study Settings section.
- 3 Click + Add.
- 4 From the list in the Parameter name column, choose f0 (Cutoff frequency).
- 5 Click Range.
- 6 In the Range dialog box, choose Number of values from the Entry method list.
- 7 In the **Start** text field, type 100.
- 8 In the **Stop** text field, type 700.
- 9 In the Number of values text field, type 51.
- IO Click Replace.
- II In the Model Builder window, click Study 3.
- 12 In the Settings window for Study, locate the Study Settings section.
- **I3** Clear the **Generate default plots** check box.
- **I4** In the **Study** toolbar, click **= Compute**.

RESULTS

Dispersion Relation

- I In the Home toolbar, click 🚛 Add Plot Group and choose ID Plot Group.
- 2 In the Settings window for ID Plot Group, type Dispersion Relation in the Label text field.
- 3 Click to expand the Title section. From the Title type list, choose Manual.
- 4 In the Title text area, type Dispersion Relation Diagram.
- 5 Locate the Data section. From the Dataset list, choose Study 2/ Parametric Solutions 1 (sol3).

- 6 Locate the Plot Settings section.
- 7 Select the x-axis label check box. In the associated text field, type f0 (Hz).
- 8 Select the y-axis label check box. In the associated text field, type real(kz) (rad/m).
- 9 Locate the Legend section. Clear the Show legends check box.

Global I

- I Right-click Dispersion Relation and choose Global.
- 2 In the Settings window for Global, locate the y-Axis Data section.
- **3** In the table, enter the following settings:

Expression	Unit	Description
if(abs(imag(acpr.kz))/abs(acpr.kz) < 1e-3,	rad/m	
abs(real(acpr.kz)), NaN)		

- 4 Locate the x-Axis Data section. From the Axis source data list, choose Inner solutions.
- 5 From the Parameter list, choose Expression.
- 6 In the **Expression** text field, type f0.
- 7 Click to expand the Coloring and Style section. Find the Line style subsection. From the Line list, choose None.
- 8 From the Color list, choose Black.
- 9 Find the Line markers subsection. From the Marker list, choose Circle.

Global 2

- I Right-click Global I and choose Duplicate.
- 2 In the Settings window for Global, locate the Coloring and Style section.
- 3 From the Color list, choose Red.
- 4 Find the Line markers subsection. From the Marker list, choose Asterisk.
- 5 Locate the Data section. From the Dataset list, choose Study 3/ Parametric Solutions 2 (sol56).
- 6 In the Dispersion Relation toolbar, click 💽 Plot.

The dispersion relation diagrams should look like the ones in Figure 4.

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